Stabilization and Performance Improvement of Power-Hardware-in-the-Loop Test Using a Robust Data-Driven Method

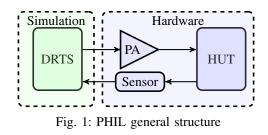
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Abstract—Power-Hardware-in-the-Loop (PHIL) setups have gained high importance in validation of the performance of newly developed instruments and devices with low risk and implementation cost. However, the interconnection of a power hardware with a simulated model via a feedback loop using measured signals may make the closed-loop system unstable. In this paper, a systematic method is proposed to design a digital filter such that the closed-loop stability of the PHIL setup is guaranteed and its performance is optimized. The design of the filter is formulated as a specific controller synthesis problem that minimizes the deviation in measured feedback current and ensures a certain robustness margin. The design problem is written as a convex optimization and solved efficiently using available solvers. Due to the data-driven characteristic of the proposed method, there is no need for the model of the Hardware Under the Test (HUT) to guarantee stability and performance. Since the design is done in a systematic fashion using advanced control design techniques with a rigorous mathematical guarantee for stability, a high-performance filter can be designed with no need for laborious manual tuning that may lead to low-performance filters. This method is also extended to a multi-scenario case including different combinations of hardware and simulated systems. The PHIL experimental results validate the effectiveness of the proposed method while satisfying the required stability margin and improving the performance significantly compared to conventional filters.

Index Terms—Power-hardware-in-the-loop, data-driven control, robust control, stabilization, convex optimization.

I. INTRODUCTION

ARDWARE-in-the-Loop (HIL) test is a flexible and efficient method to study the behavior of an instrument (i.e. HUT) and its interactions with the environment by measuring its states. HIL tests are usually conducted when a part of the system needs to be scaled-up [1] or when the objective is to study various normal and/or extreme and rare conditions [2]. The main objective of a HIL test is to be able to measure the states of the instrument with high fidelity. Thus, it is an important factor that the measurement data is not deviated. However, the most important factor is to make the



system stable because in an unstable system no measurement is possible and the instruments may also be damaged. Since the measured signal is used in the feedback loop in the HIL setup, any filtering and process on this signal can directly affect the stability of the system. The main challenge is to condition, filter and process the measured signal such that the system remains stable while the quality of measurement data is not compromised.

A HIL setup can be seen as a complex measurement setup which is composed of different subsystems. When the environment is simulated by means of a Digital Real-Time Simulator (DRTS), different operating conditions can be tested with minimum cost and risk. If the HUT interacts with the DRTS via signals with no (or negligible) real power exchange and the controller is realized as the HUT, the HIL setup is called Control-Hardware-in-the-Loop (CHIL) [3].

In power systems, when a new component is added, there is a need for a series of tests to ensure the resiliency of the system. These tests should not impose high risk and cost. Using PHIL setups, different operation scenarios can be studied with reduced cost and time [4]. Moreover, in a tightly controlled laboratory environment, a PHIL setup can facilitate testing conditions which may be very extreme and/or dangerous in a power system [5]. In most applications, a PHIL setup is a HIL setup that includes DRTS and HUT which exchanges a considerable amount of power with the environment (e.g. when a real battery should be charged with a simulated synchronous generator) [6], [7]. In a PHIL setup, an interface between DRTS and HUT is required which basically includes a Power Amplifier (PA) and some sensors. PA can be realized either as a voltage amplifier or a current amplifier depending on the structure of the PHIL setup. However, in both cases, it is common to be denoted as PA in the PHIL literature. The general structure of a PHIL setup is shown in Fig. 1. In PHIL setups, DRTS should run with short and deterministic

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time steps which means that the time elapsed in the simulation is exactly equal to the time measured by a real-world clock [4]. In an ideal case, the PA and sensors should act as fixed gains in all frequencies with no delay and the simulation step time in DRTS should be zero, which are not realizable in practice [8], [9]. Moreover, depending on the combination of the system in simulation and HUT, the open-loop structure of the system may not necessarily be strictly proper which may result in high gain at high frequencies. The combination of the delay and the latter property may end up in PHIL instability that can put the HUT and PA under severe stress and cause damage [10]. As a remedy, the loop frequency response can be attenuated (more importantly at high frequencies) such that the closedloop system becomes stable. It can be realized by adding lowpass filters on measured signals, simulated impedances inside DRTS, physical impedances or a combination of those. However, these changes may result in the deviation of the measured value from what it should truly be. Different implementation methods and structures (a.k.a Interface Algorithms (IA)) are proposed to provide stability while trying to keep measurement signal deviation low. Some of the most well-known IAs are Ideal Transformer Model (ITM) [11], Time-variant First-order Approximation (TFA) [12], Transmission Line Model (TLM) [11], Partial Circuit Duplication (PCD) [11] and Damping Impedance Method (DIM) [10]. Among these methods, PCD, DIM and TLM are essentially based on the idea of using additional physical impedances which is not practically desirable because it is harder to implement, less flexible and has higher maintenance costs specifically in highpower applications [13]. Moreover, when the added hardware includes inductance, the performance in high frequencies is considerably affected [14]. TFA does not require an extra impedance, however, it has numerical stability problems and high sensitivity to noise [10]. The performance of some of the interface algorithms and their impact on the stability margin are compared in [15]. Among the mentioned methods, ITM is the best method in terms of implementation and is widely used in practice [9], [16]–[21]. However, stabilization is the main challenge using this method [10].

One of the main contributions of this paper is that the closed-loop stability of the PHIL setup is guaranteed without any knowledge about the parameters of the hardware. There are two important points, namely, stability guarantee and data-driven property.

Stabilization is the primary goal of almost all of the proposed methods in this area due to its safety hazards for the operators and the economic costs. Consequently, closedloop stability should be guaranteed in the design process. However, in contrast to most of the proposed methods, it is not enough to guarantee stability only for the nominal system. Uncertainty, as one of the main characteristics of PHIL systems, should also be considered in the proposed solution. The simulated system may not only have a single operating condition but instead a set of different possible conditions. For example, a power grid may have different typologies based on the status of the breakers or the HUT can have different possible modes which can be addressed by considering multimodel uncertainty. Moreover, the behaviour of the system may slightly and gradually change during the operation. For example, the resistive part may slightly change during a test which can be dealt with by a proper stability margin.

Using the measurement data directly for the design and bypassing the modeling step is another important characteristic of the proposed method. Conventionally, the data was used to construct a model (a.k.a system identification [22]) by an optimization and based on that model the controller was designed by another optimization. However, it should be assumed that the structure of the system is known (e.g. order) and a good low-order model can be obtained from the data which is not necessarily a valid assumption in practice. Specifically, in PHIL applications, it is not realistic to assume the availability of a reliable and accurate low-order model of the system because the unavailability of such a model is the major motivation behind conducting a PHIL experiment [13]. As an alternative, in direct-data-driven methods, the controller is designed directly from the data via only one optimization. In these methods, there is no need to put any assumption on the structure of the system which makes it suitable for PHIL applications. Basic direct-data-driven methods are reviewed in [23]. Recently, thanks to the advancements in solver technology and computation power, these methods have gained more attention [24]–[33]. The data used for synthesizing the controller may be in time-domain [24]–[26] or in frequency-domain [27]–[33]. One of the advantages of the latter is that the closed-loop sensitivity functions can be shaped directly in the frequency domain. However, in the application of controller design for PHIL setup, in addition to closed-loop functions, the controller itself should also be shaped which is not easily possible with the data-driven methods that are proposed in the literature.

Conventionally, in PHIL applications a parametric model (based on physical parameters of the system or identifying a low-order parametric model) of each element in the loop, such as PA, HUT, sensors, etc. are generated separately [34]-[36]. There are a limited number of researches addressing this problem by trying to use the data of PHIL systems [34], [37]–[41]. In [37]–[39], authors use the measured data and perform parametric identification of the input impedance of HUT. In [40], a similar method for impedance identification is used and a passivity criterion is considered for providing stability. However, for stabilization, the DIM method is used which has its own limitations as mentioned earlier. In [34], by considering the general structure of the system as a resistive voltage divider, it is assumed that the elements in the PHIL setup can be approximated by a first-order parametric model and the delay is approximated for a PHIL setup and then the stability is checked by the Routh criterion. Apart from the error introduced from the low order assumption and approximation of the delay, the studied structure is limited to a resistive voltage divider along with a very basic compensator that leads to low performance.

An efficient way to solve the design problem is to formulate it as a convex optimisation problem because there are very well-established numerical methods for convex programming which can handle a problem with a large number of variables and constraints [42]. Moreover, there are advanced solver technologies available to solve this type of problems (e.g.

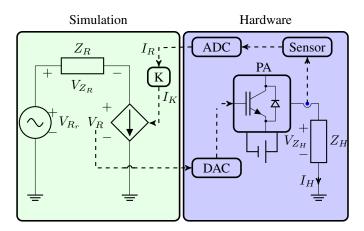


Fig. 2: PHIL circuit with ITM interface algorithm

[43]). Consequently, the main challenge is not to solve the convex programming problem, but instead, to formulate the practical problem at hand as a convex programming problem. To the best knowledge of the authors, there is no published literature that proposes a systematic, flexible and comprehensive controller design method for PHIL setups that guarantees closed-loop stability based on the data of the system without any assumption on the system structure (order, delay, etc.). In this paper, the problem of designing a high-performance filter that guarantees the closed-loop stability of a PHIL system is converted to a specific data-driven controller design method using convex optimization which can be solved fast and efficiently. It should be mentioned that the objective here is to put some constraints on the controller (filter) instead of the closed-loop transfer function which is usually done in the classical controller design methods. Only the frequency response of the loop is required to optimize the controller (filter) parameters under a predefined robust stability margin constraint. The proposed method is used to design a robust controller for the multi-scenario case as well.

The rest of the paper is organized as follows: In Section II, the structure of the PHIL setup is defined and in Section III, the controller design method using convex optimization is detailed. In Section IV, the process of data measurement and controller design for a case study is explained and finally in Section V, some concluding remarks are mentioned.

II. PHIL SYSTEM STRUCTURE

ITM can be implemented using a current source or a voltage source on the PA side which are dual structures. In this paper, the voltage source ITM (simply denoted as ITM) is used and its general structure is depicted in Fig. 2, which is similar to the structure used in [4], [9], [10], [13], [21], [44], [45]. The results can be used straightforwardly for the current source ITM as well. The Thévenin theorem is applied in PHIL system analysis as the key modelling principle [46] and linear approximation of the system at different operating points enables the use of linear control system theory [4]. In fact, linear assumption and using linear techniques are common in the literature (e.g. [4]–[6], [13], [15], [15], [38], [44], [46]– [48]). However, multi-model structures or Linear Parameter Varying (LPV) methods [31], [49] can be used in some cases if this assumption is not valid. In the structure shown in Fig. 2, V_{R_r} and Z_R are equivalent voltage and impedance of the grid simulated in DRTS. This part is connected in series to a controlled current source which reflects the image of the current in the hardware side to the DRTS side. On the other hand, the voltage over the controlled current source (i.e. V_R) is amplified by PA to generate voltage V_{Z_H} , and, Z_H represents the HUT.

Feedback current filtering is a well-known solution for stabilizing the closed-loop system [14], [45]. In this method, K as an extra element is added to the loop in order to make it stable, while it should not deteriorate the signals and in an ideal case, it should act as a unity gain. In fact, the more K is different from the unity the more deviation is imposed on the PHIL results. Besides, in the ideal case, PA should generate exactly what is asked for (i.e. $V_{Z_H} = V_R$). However, it does not happen due to the delay and non-ideal elements. The PHIL block diagram is shown in Fig. 3. In this figure, G_P represents the transfer function of the PA while its input is the reference voltage and its output is V_{Z_H} which includes the delay and usually has a low pass characteristic. G_H represents the transfer function corresponding to Z_H with its voltage and current as the input and output, respectively. G_S denotes the transfer function of the current sensor. K is the controller (filter) which acts on the feedback current in order to stabilize the closed-loop system. G_R is the transfer function corresponding to Z_R with the input I_K and the output V_{Z_R} . DRTS is run at step T_R which is limited by the DRTS technology and the complexity of the simulated system. Since the simulation in DRTS is in discrete-time and the hardware is in continuous-time, there is a need for ADC and DAC to connect these parts, which in the block diagram can be seen as a Zero-Order-Hold (ZOH) block and a sampler. One common method to assess the stability of the system is to model each part of the system separately and determine a simplified transfer function for the whole loop where some blocks may be ignored. For example, the dynamics of PA is ignored in [10] or modelled as a mixture of delay and a low-pass filter in [46]. For G_H , the first principles modeling (based on the assumption of known parameters and known physical equations governing the system) is common which is prone to uncertainty due to unknown parameters and physical interactions. Moreover, in many cases, the system is only analyzed in continuous-time ignoring the impact of the ZOH and the sampler. The mentioned issues may lead to an invalid understanding of the loop behavior which can end up in an incorrect stability analysis.

In this paper, a fully data-driven controller design method is proposed where the measured data is directly used for the design with no parametric identification step. To do so, the frequency response of the system from V_R and to I_R can be computed by applying a sine-sweep or a Pseudo-Random Binary Sequence (PRBS) to V_R . This system is denoted as G_{IH} which includes the ZOH, G_P , G_H , G_S and the sampler. Since G_R is known from the simulation, the total

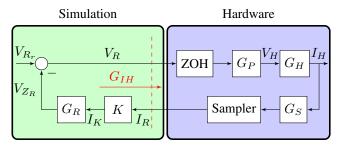


Fig. 3: PHIL setup block diagram

plant frequency response, G, can be considered as a series connection of G_{IH} and G_R .

III. CONTROLLER (FILTER) DESIGN

In this section, a data-driven controller design method is introduced to design a low-pass filter for a PHIL system that guarantees the closed-loop stability and performance of the system in terms of accuracy and disturbance rejection. For the practical ease of implementation, the controller (filter) is defined in discrete-time, however, since the design method is in the frequency-domain, it can readily be rewritten for continuous-time controllers as well.

The system to be controlled is a Linear Time-Invariant (LTI) single-input single-output system. It is assumed that the frequency response of the system is computed based on a finite set of sampled measurement data and represented by a transfer function $G(e^{j\omega}) \in \mathbb{C}$ and is available for all $\omega \in \Omega = [-\omega_n, \omega_n]$, where $\omega_n = \pi/T_s$ is the Nyquist frequency and T_s is the sampling period.

A. Performance

As mentioned previously, K is an added element to the system to make the closed-loop system stable. However, it should not change the results of the experiment in an ideal case. Assume that the relative error between I_K and I_R is defined as follows:

$$\frac{I_K - I_R}{I_R} = K - 1 \tag{1}$$

which has a close connection to the ripple concept in filter design (see [50]). Consequently, the design problem can be defined as follows:

$$\min_{K} \|K - 1\|_{2}^{2} \tag{2}$$

Note that, this type of objective is not common in the classical controller design methods, where usually a norm of a closed-loop transfer function is minimized. Therefore, a standard \mathcal{H}_2 or \mathcal{H}_{∞} controller design method cannot be used here. Using the definition of the two-norm of a transfer function, it can be shown that the problem (2) is equivalent to

$$\min_{K} \int_{-\omega_{n}}^{\omega_{n}} \left| K(e^{j\omega}) - 1 \right|^{2} d\omega \tag{3}$$

In order to have an upper bound on the gain of the filter, the following constraint can be used:

$$|K(e^{j\omega})| < W_u(\omega) \quad \forall \omega \in \Omega \tag{4}$$

Similarly, a lower bound is required to guarantee the signal passes up to a predefined value which can be written as:

$$|K(e^{j\omega})| > W_l(\omega) \quad \forall \omega \in \Omega \tag{5}$$

In addition, it is clear that the result of this optimization does not necessarily stabilize the closed-loop system. Particularly, since G is not necessarily strictly proper and may have a very large gain at high frequencies, for closed-loop stability the controller (filter) K should have frequency roll-off at high frequencies. In the following part, a constraint for stability margin will be discussed.

B. Robustness

In order to make the system robustly stable, a guaranteed stability margin is crucial. In this paper, the modulus margin (M) has been used as the stability margin which guarantees the well-known gain margin and phase margin simultaneously. Modulus margin is the minimum distance of the open-loop transfer function from the critical point (i.e. -1 + 0j) in the Nyquist diagram, which can be defined as:

$$M = \left(\sup_{\omega} |\mathcal{S}(e^{j\omega})|\right)^{-1} \tag{6}$$

where $S = (1 + L)^{-1}$ is the output sensitivity function and L = GK is the open-loop transfer function. For a closed-loop stable system, the modulus margin can be guaranteed by:

$$|\mathcal{S}(e^{j\omega})| < M_l^{-1} \quad \forall \omega \in \Omega \tag{7}$$

where M_l is the desired lower-bound for the modulus margin M. The above constraint guarantees a minimum distance from the critical point in the Nyquist diagram but it does not guarantee that the Nyquist diagram will not turn around the critical point. This problem is studied at the end of this section.

C. Disturbance Rejection

In addition to the previous constraints, the output disturbance rejection can also be considered as a performance in the controller design. The transfer function from the output disturbance to I_K is $\mathcal{U} = K(1+L)^{-1}$, where \mathcal{U} is the input sensitivity function and its magnitude can be limited using the following constraint:

$$|\mathcal{U}(e^{j\omega})| < U_u \quad \forall \omega \in \Omega \tag{8}$$

where U_u is the desired upper bound for the input sensitivity function.

D. Controller Design Problem

The controller design problem can be cast as the following nonlinear constrained optimization problem:

$$\min_{K} \int_{-\omega_n}^{\omega_n} (K-1)(K-1)^* d\omega$$
 (9a)

s.t.
$$KK^* < W_u W_u^* \quad \forall \omega \in \Omega$$
 (9b)
 $KK^* > W_u W_u^* \quad \forall \omega \in \Omega$ (9c)

$$KK > W_l W_l \quad \forall \omega \in \Omega \tag{9c}$$
$$SS^* < M_l^{-2} \quad \forall \omega \in \Omega \tag{9d}$$

$$\mathcal{U}\mathcal{U}^* < U_u^2 \quad \forall \omega \in \Omega. \tag{9d}$$

In this optimization problem, the objective (i.e. (9a)) is equivalent to (3). The first and the second constraint (i.e. (9b) and (9b)) are equivalent to (4) and (5), respectively. The third constraint (i.e. (9d)) is equivalent to (7) and provides a predefined stability margin. And finally, the fourth constraint (i.e. (9e)) is equivalent to (8) and is added to limit the magnitude of the transfer function from the disturbance to the I_K in order not to let the measurement noise and disturbance be amplified in the simulation. Basically, from the control point of view, the input and output closed-loop sensitivity functions (i.e. S and U) act toward disturbance rejection and help the quality of the signals not to be affected by disturbance and noise. For the sake of simplicity in notation, the functions' arguments are omitted in the above problem and in the rest of the paper and are only reiterated when deemed necessary. This problem is a non-standard control problem, non-convex and with an infinite number of constraints. In the sequel, the steps required to change this problem into a convex optimization are explained.

1) Controller Structure and problem formulation: The controller is defined as:

$$K(z) = \frac{X(z)}{Y(z)} \tag{10}$$

where both X and Y are polynomials in z as follows:

$$X(z) = (x_0 + x_1 z + \dots + x_{n_K} z^{n_K})$$
(11a)

$$Y(z) = (y_0 + y_1 z + \ldots + y_{n_K - 1} z^{n_K - 1} + z^{n_K})$$
(11b)

where x_i for $i = 0, 1, ..., n_K$ and y_i for $i = 0, 1, ..., n_K - 1$ are the controller parameters. Let Q = X - Y and P = Y + GX, then obviously

$$K-1 = \frac{Q}{Y}, \ \mathcal{S} = \frac{1}{1 + \frac{GX}{Y}} = \frac{Y}{P}, \ \text{and}, \ \mathcal{U} = \frac{\frac{X}{Y}}{1 + \frac{GX}{Y}} = \frac{X}{P}.$$

Consequently, problem (9) can be written as:

$$\min_{X,Y} \int_{-\omega_n}^{\omega_n} (\frac{Q}{Y}) (\frac{Q}{Y})^* d\omega$$
(12a)

s.t.
$$\left(\frac{X}{Y}\right)\left(\frac{X}{Y}\right)^* < W_u W_u^* \quad \forall \omega \in \Omega$$
 (12b)

$$(\frac{X}{Y})(\frac{X}{Y})^* > W_l W_l^* \quad \forall \omega \in \Omega$$
(12c)

$$(\frac{Y}{P})(\frac{Y}{P})^* < M_l^{-2} \quad \forall \omega \in \Omega \tag{12d}$$

$$(\frac{X}{P})(\frac{X}{P})^* < U_u^2 \quad \forall \omega \in \Omega$$
 (12e)

This problem is still non-convex and its convexification is explained in the next part.

2) *Convex Controller Design Problem:* It can be shown that (12a) is equivalent to:

$$\min_{X,Y} \int_{-\omega_n}^{\omega_n} \gamma(\omega) \tag{13a}$$

s.t.
$$(\frac{Q}{Y})(\frac{Q}{Y})^* < \gamma(\omega) \quad \forall \omega \in \Omega$$
 (13b)

where $\gamma(\omega) > 0$ for all $\omega \in \Omega$ is an unknown function. Moreover, as a general procedure, constraints of the problem (12) are written in the convex-concave form and then, the concave part is linearized around an initial stabilizing controller $K_c(z) = X_c(z)Y_c^{-1}(z)$ to obtain a convex constraint. To start with (12b), it can be written as:

$$XX^* - W_u YY^* W_u^* < 0 \quad \forall \omega \in \Omega.$$
⁽¹⁴⁾

It should be noted that left hand side of (14) has quadratic from in X and Y and according to (11), X and Y are linear in controller parameters (i.e. x_i for $i = 0, ..., n_K$ and y_i for $i = 0, ..., n_K - 1$).

Let $F := W_u Y$ and $F_c := W_u Y_c$. By developing $(F - F_c)(F - F_c)^* \ge 0$, it can be shown that:

$$XX^* - F_cF^* - FF_c^* + F_cF_c^* \le 0 \quad \forall \omega \in \Omega$$
 (15)

is a sufficient convex constraint for (14). Similarly, other constraints in the problem can be convexified.

3) Semi-definite Controller Design Problem: In general, the frequency response that is obtained from the sampled data can be calculated as [51]:

$$G(e^{j\omega}) = \frac{\sum_{k=0}^{N-1} y(k) e^{-j\omega T_s k}}{\sum_{k=0}^{N-1} u(k) e^{-j\omega T_s k}}$$
(16)

where u(k) is the input and y(k) is the output which are assumed equal to zero for $k \ge N$ and k < 0. As it can be seen, this function is continuous in frequency and can be calculated at any arbitrary frequency. If the plant obtained in (16) is used to construct the problem in (12), it will be a semi-infinite programming (i.e. with an infinite number of constraints). In order to make the problem tractable, the frequency response can be sampled at a finite set of frequencies, $\Omega_N = \{\omega_1, \ldots, \omega_N\}$ (since the constraints are symmetric, non-negative frequencies are considered). As a result, the semi-infinite problem would be changed to a tractable semi-definite programming. Following the same convexification procedure for all of the constraints, the controller design problem can be written as a semi-definite programming as follows:

$$\min_{X,Y} \sum_{\Omega_N} \gamma(\omega) \tag{17a}$$

s.t.
$$Q\gamma^{-1}(\omega)Q^* - Y_cY^* - YY_c^* + Y_cY_c^* < 0$$
 (17b)

$$XX^{+} - F_{c}F^{+} - FF_{c}^{+} + F_{c}F_{c}^{+} < 0$$
 (1/c)

$$W_l Y Y^* W_l^* - X_c X^* - X X_c^* + X_c X_c^* < 0 \qquad (1/d)$$

$$YY^*M_l^2 - P_cP^* - PP_c^* + P_cP_c^* < 0$$
 (17e)

$$XX^*U_u^{-2} - P_cP^* - PP_c^* + P_cP_c^* < 0$$
(17f)

for all ω in Ω_N where $P_c = Y_c + GX_c$. In this algorithm, $\gamma(\omega_1)$ to $\gamma(\omega_N)$ are N positive dummy variables that make the objective function linear. Constraint set (17b) using Schur complement lemma has convex form and constraint sets (17c)-(17f) have quadratic-convex from. Consequently, this problem has a linear objective function with convex constraints which is a convex programming and can be solved efficiently and fast by available convex-optimization solvers. Problem (17) is obtained based on the convexification of the original problem around the initial controller. Consequently, even though the stability will be kept anyway, the performance will highly depend on the choice of the initial controller. To improve the performance an iterative approach can be used that considers the optimal controller from the previous iteration as the initial controller for the next iteration. This way, the stability property will be preserved while the performance will converge to that of a local minimum or a saddle point of the original nonconvex problem [30].

4) Closed-loop stability: If $P_cP^* + PP_c^* > 0$, $Y \neq 0$ and $Y_c \neq 0$ for all $\omega \in \Omega$ and the order of Y equals the order of Y_c , then the number of encirclements of GK around the critical point will be equal to that of GK_c (see Theorem 2 in [30] for multivariable systems). It can be shown that (17e) or (17f) ensures that $P_cP^* + PP_c^* > 0$ and (17b) is sufficient for $Y \neq 0$ and $Y_c \neq 0$. Consequently, the closed-loop system will be stable if the initial controller K_c is stabilizing and the order of Y equals the order of Y_c . It should be noted that assuming the availability of an initial stabilizing controller is not a strong assumption for stable plants since a sufficiently small gain always stabilizes the output-feedback closed-loop system.

E. Controller Design for Multi-Scenario case

In this section, the problem of encountering different scenarios in the PHIL setup is addressed. For example, when different topologies of a grid or testing more than one type of HUT are considered. All these scenarios may amount to different Z_R and Z_H which eliminates all the stability and performance guarantees. As the number of scenarios increases, designing without a systematic method becomes more time consuming and may lead to an unnecessary conservative result. Assume there are n_G scenarios each of which corresponds to one PHIL setup configuration (i.e. one (Z_R, Z_H)). The set of all models is defined as $\mathcal{G} = \{G_1, G_2, \ldots, G_{n_G}\}$ where G_i denotes frequency response corresponding to *i*-th scenario. Consequently, (17) can be extended to:

$$\min_{X,Y} \sum_{\Omega_N} \gamma(\omega) \tag{18a}$$

s.t.
$$Q_i \gamma^{-1}(\omega) Q_i^* - Y_c Y^* - Y Y_c^* + Y_c Y_c^* < 0$$
 (18b)

$$XX^* - F_c'F^{**} - F'F_c'^* + F_c'F_c^{**} < 0$$
(18c)

$$W_l Y Y^* W_l^* - X_c X^* - X X_c^* + X_c X_c^* < 0$$
 (18d)

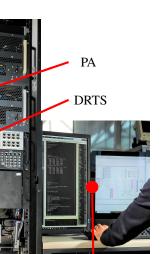
$$YY^*M_l^2 - P_{ci}P_i^* - P_iP_{ci}^* + P_{ci}P_{ci}^* < 0$$
 (18e)

$$XX^*U_u^{-2} - P_{ci}P_i^* - P_iP_{ci}^* + P_{ci}P_{ci}^* < 0$$
 (18f)

for all ω in Ω_N and for all G_i in \mathcal{G} .

IV. CASE STUDY

The controller design problem explained in Section III is applied on a PHIL setup. In this case study, Opal-RT OP5600 and Puissance plus AC amplifier PA-3x1000-AB/260V-7.7A are used as DRTS and PA, respectively. The PHIL test setup is shown in Fig. 4. It should be noted that the results presented in this section do not aim to reflect the characteristics of any specific system or device. In this case study, resistiveinductive impedances for both Z_R and Z_H are selected which is common in PHIL system studies (e.g. [6], [21], [44], [52]). For the base-case scenario, $L_R = 9$ mH, $L_H = 6$ mH, and $R_R = R_H = 11\Omega$. It should be emphasized again that these



User Interface

Protection

HUT

Host Computer

Fig. 4: PHIL setup

parameters are used neither for the stability analysis nor for the design and are just mentioned for the completeness of the representation.

A. Data Measurement

A PRBS signal with order 10 and magnitude of 10V, which is generated in DRTS is used to excite the system and the response of the system is logged. In practice, the measurement data is contaminated with noise whose impact on the frequency response can be considered with an additive uncertainty as $\tilde{G}(e^{j\omega}) = G(e^{j\omega}) + W_u(e^{j\omega})\Delta$ where $W_u(e^{j\omega})$ is the uncertainty filter and Δ is the uncertainty disk [53]. W_{u} can be calculated using an estimate of the noise spectrum from the input-output data [22]. A very practical method to reduce the impact of the noise is to use periodic signals [51]. In this case study, 4 periods of measured data are used to calculate the frequency responses. In Fig. 5, one period of the excitation signal and the measured I_R are shown. It can be seen that based on the order of the PRBS signal and the sampling time of DRTS, the whole data measurement process takes less the one second. The Fourier transform of input and output data is calculated and G_{IH} is obtained. Since Z_R is known, its frequency response and consequently the frequency response of the plant G can be obtained as shown in Fig. 6. In Fig. 7, the Nyquist diagram shows one encirclement of the critical point, meaning that without a controller (filter) the PHIL becomes unstable.

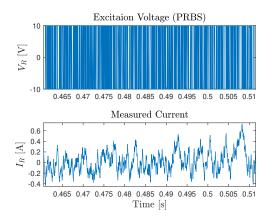
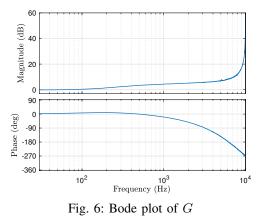


Fig. 5: Excitation at V_R and measured I_R



B. Controller (filter) Design

Finding a stabilizing controller for this system is very easy since even a small gain provides closed-loop stability. However, this is not practical since for $K \ll 1$, the current $I_K \ll I_R$ and the precision would be very low. In the literature, a common practice is to use a simple first-order ([9], [13], [19], [20], [44]–[47]) or a second-order Butterworth ([5], [48]) low-pass filter for PHIL stabilization.

As the first step, the limitations of the conventional lowpass filtering method is studied to see what can be achieved using the simple filters proposed in the literature. Assume that a set of first and second-order low-pass Butterworth filters

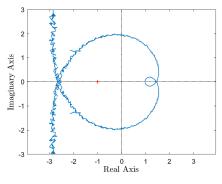


Fig. 7: Nyquist plot of G

TABLE I: Controller Design parameters

Param.	Value	Freq. [Hz]	W_l
M_l	0.5	[0,50]	0.9999
N	511	(50,300]	0.999
U_u	-3dB	(300,900]	0.99
n_K	17	(900,1800]	0.9
W_u	$2 - W_l$	$(1800, 1/2T_s]$	0

are available with bandwidth varying from 0.5 kHz to 2 kHz with steps of 0.5 kHz. The Nyquist plots of the open-loop transfer functions using these filters are shown in Fig. 8a. It can be seen that for both first-order and second-order filters the Nyquist diagram approaches the critical point as the bandwidth of the filter increases. In order to assess the stability margin and output disturbance rejection in the control signal, the bode diagrams of the output and input sensitivity functions are depicted in Fig. 8b and Fig. 8c, respectively. As shown in Fig. 8, there is a trade-off between the bandwidth of the controller and the stability margin. In addition, it is very important to note that simply increasing the order of the filter affects the stability margin adversely. Consequently, the stability issue is even more critical when the order is increased (e.g. see Fig. 8a and Fig. 8b). In other words, when there is no systematic method to take advantage of the added capability and when the added complexity is not dealt with correctly, it may end up in the system's instability. The lack of a systematic, efficient, and, advanced design method may be the main reason why in the current literature the order is kept low. Considering the mentioned items, a first-order filter (which was used also in [9], [13], [19], [20], [44]-[47]) with a bandwidth of 1250 Hz, denoted by K_0 , is selected and compared with our controller in the sequel.

For designing the controller, the problem (17) is solved using MOSEK [43] in less than 20 seconds on a conventional desktop computer. A fast systematic design method is very valuable, compared with the manual tuning when a large number of tests should be carried out in a short time. The design parameters are listed in Table I and the designed controller is denoted as K_D . The Nyquist diagram of systems using K_0 and K_D are compared in Fig. 9. It can be seen that both controllers have made the closed-loop system stable. However, K_D has provided a farther distance from the critical point, which results in higher robustness (M = 0.5 for K_D vs. M = 0.39 for K_0). The disturbance rejection performance of the closed-loop systems are compared in Fig. 10. As shown in the figure, the input sensitivity function has not violated the required limit and provides better disturbance attenuation compared to K_0 . Finally, the frequency response of K_D is compared with that of K_0 in Fig. 11, which shows considerable improvement. Fig. 12 clearly shows the impact of the constraints on the controller. In order to have a better overview, the regions satisfying different accuracies are listed in Table II. It can be seen that requirement set by the constraints are met and the performance compared to the conventional filter are improved significantly. To validate the performance of the proposed method, the designed controller is implemented on the PHIL setup and a sinusoidal sweep signal is applied as

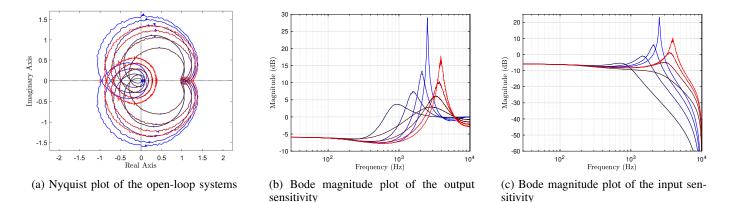


Fig. 8: Systems using second-order (blue) and first-order (red) filters (dark to light represents 0.5 KHz to 2 kHz bandwidth)

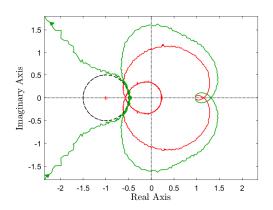


Fig. 9: Nyquist plot of the open-loop systems using K_0 (red) and K_D (green), (corresponding constraint (dashed-black))

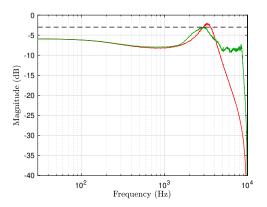


Fig. 10: Bode magnitude plot of the input sensitivity for systems using K_0 (red) and K_D (green), (corresponding constraint (dashed-black))

 V_{R_r} . Fig. 13 compares I_R and I_K at different frequencies. In the fundamental frequency (i.e. 50 Hz), K_D almost perfectly tracks while K_0 has a slight deviation. In 300 Hz, the deviation using K_0 is visible. The difference at 900 Hz is considerable while at 1800 Hz, the response of K_0 is significantly deviated.

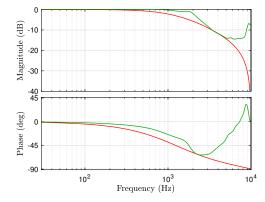


Fig. 11: Bode plot of the controllers K_0 (red) and K_D (green)

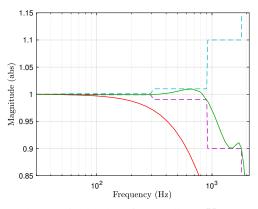


Fig. 12: Bode magnitude plot of controllers (K_D (green), K_0 (red), W_l (dashed magenta), W_u (dashed cyan))

TABLE II: Frequency regions for different error bounds

$\max K-1 $	Up to Freq.	Up to Freq.	Up to Freq.	
	(K_0)	(Expected)	(realized by K_D)	
	[Hz]	[Hz]	[Hz]	
0.0001	8	50	67	
0.001	55	300	380	
0.01	177	900	900	
0.1	601	1800	1800	

TABLE III: Multi scenario system parameters (L_R [mH], L_H [mH], R_R [Ω], R_H [Ω])

Sc.	Param.	Sc.	Param.	Sc.	Param.	Sc.	Param.
1	(3,3,11,22)	5	(3,3,11,11)	9	(3,6,11,22)	13	(3,6,11,11)
2	(9,3,11,22)	6	(9,3,11,11)	10	(9,6,11,22)	14	(9,6,11,11)
3	(3,3,22,22)	7	(3,3,22,11)	11	(3,6,22,22)	15	(3,6,22,11)
4	(9,3,22,22)	8	(9,3,22,11)	12	(9,6,22,22)	16	(9,6,22,11)

C. Discussion on design parameters and problem constants

Because the proposed method is systematic and data-driven, unlike most methods in the literature, it is independent of the parameters of the problem. These values are selected as typical values just to provide a showcase. In this paper, two distinct sets of fixed values can be seen.

1) System constants: These constants stem from the PHIL equipment and are not selected by the controller designer. For example, T_s depends on the computation power of the DRTS and is selected to be 50 μ s which is a typical value [54]. As another example, L_R and R_R depend on the simulated grid and L_H and R_H are based on the HUT. In this paper, the grid is assumed to be a distribution system in which the X/R is relatively small. As the base case, the X/R- ratio of the impedances in HUT and DRTS belong to the interval [0.15, 0.26] so do the lines 4c-70 and 4c-95-SAC-AC of the IEEE of European low voltage test feeder [55].

2) Design parameters: These parameters are selected by the controller designer based on the expected performance. For example, the modulus margin is selected based on the required robustness and is selected to be $-6 dB \approx 0.5$ which is a typical value in the controller design. N is the number of sampled points in the frequency-domain which can be chosen relatively large without severe impact on the solver time [30]. U_u can be selected based on the measurement noise level and is simply chosen to be $-3 dB \approx 0.7$). If the measurement is highly noisy, this value can be set lower. n_K determines the flexibility of the controller, and in this problem, with $n_K = 17$ a good performance has been achieved. W_l and W_u determine maximum deviation in the measured current and for the sake of simplicity, are set to be symmetric. In this paper, it has been required that |K-1| must be less than 0.0001 up to the first harmonic, less than 0.001 up to the 6th harmonic, less than 0.01 up to the 18th harmonic, and, less than 0.1 up to the 36th harmonic. These values are selected as an example showing how the design requirements can be met by this method. In practice, these values should be selected based on the desired performance of the problem.

D. Multi-Scenario Case

In this part, 16 different scenarios are considered whose corresponding parameters are listed in Table. III and frequency responses are shown in Fig. 14. The problem (18) is solved considering all scenarios as one comprehensive programming problem and the designed controller is denoted as K_{DM} . For this design, a guarantee for |K - 1| to be less than 0.001 up to the first harmonic, less than 0.01 up to the 6th harmonic, and, less than 0.1 up to the 12th harmonic is considered. The Nyquist diagram of systems using the K_0 and K_{DM} are

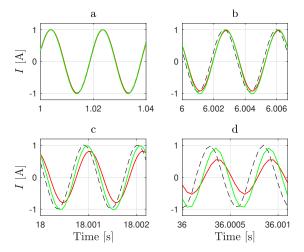


Fig. 13: Current measured from PHIL setup (I_R (black dashed), I_K with K_D (solid green), I_K with K_0 (solid red)) a, b, c, and d, correspond to 50 Hz, 300 Hz, 900 Hz, and 1800 Hz, respectively.

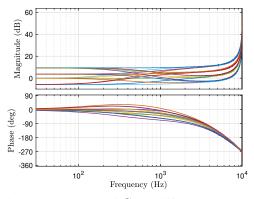


Fig. 14: Bode plot of G_i in different scenarios

compared in Fig. 15 which shows that K_0 cannot stabilize all scenarios, while K_{DM} stabilizes all scenarios and provides M = 0.5 for all of them. Fig. 16, shows that K_{DM} has satisfied the required attenuation for all scenarios. If one wants to use the conventional method to design a controller which stabilizes all models with comparable robustness and disturbance rejection (named as K_{0M}), the bandwidth should be decreased to 400 Hz, which results in losing performance. In Fig. 17, the bode diagram of K_{0M} is compared with K_{DM} , K_D and K_0 . As expected, the performance of K_{DM} is limited compared to K_D since challenging scenarios are included (e.g. scenario No. 8 and scenario No. 6). However, compared to the K_{0M} , it shows considerable improvement. As an example, scenario No. 8 is tested in PHIL setup using K_D and K_{DM} and the results are shown in Fig. 18. As expected, K_0 destabilizes the system while K_{DM} stabilizes and shows good performance.

V. CONCLUSION

In this paper, the problem of PHIL setup stabilization and performance improvement has been addressed and a datadriven controller design method has been proposed. In this method, in order to reduce the impact of the added controller

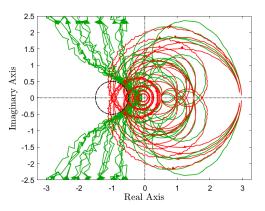


Fig. 15: Nyquist plot of open-loop systems corresponding to different scenarios by using K_0 (red) and K_{DM} (green)

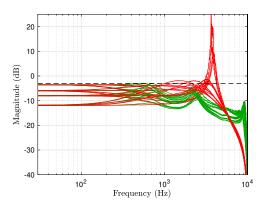


Fig. 16: Bode magnitude plot of input sensitivity of systems corresponding to different scenarios by using K_0 (red) and K_{DM} (green)

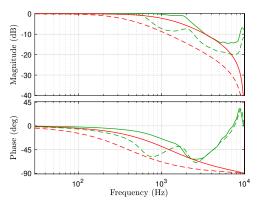


Fig. 17: Bode of the controllers, K_{0M} (dashed red), K_{DM} (dashed green), K_0 (solid red), and, K_D (solid green)

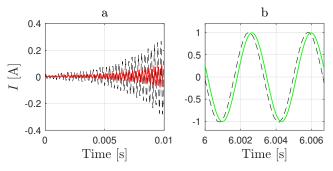


Fig. 18: I_R (dashed) and I_K (solid) corresponding to scenario No. 8 a) closed-loop instability using K_0 , b) tracking using K_{DM} at 300 Hz

on the feedback current signal, the two-norm of the difference between controller and unity is considered as the objective function. In addition, the constraints corresponding to the stability, robustness, disturbance rejection and the frequency dependent constraints corresponding to maximum allowable signal amplification and attenuation are added to the synthesis problem. These constraints are convexified by linearizing around an initial stabilizing controller, which led to quadratic convex constraints. Since the whole process of data collection and controller design takes roughly less than 30 seconds, the proposed method can be efficiently used as an automatic controller design in PHIL setups. Because the proposed method is in a data-driven framework, there is no need for knowledge about the parameters, order or even structure of the hardware. In addition, the proposed method is extended to a multiscenario case where using one controller, different topologies in HUT or simulation can be tested. The results show that the designed controller can improve the performance of the simulation significantly compared to a conventional low-pass filter. The implementation of the designed controller on the PHIL setup validates the performance of the proposed method.

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