
Iterative Tuning of Restricted-Complexity Controllers

A Perspective

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Outline

1. Introduction

- Identification for control
- Iterative Feedback Tuning

2. Iterative Correlation-Based Controller Tuning

- Correlation approach using Instrumental Variables
- Convergence and Frequency Analysis
- Simulation and Experimental Results

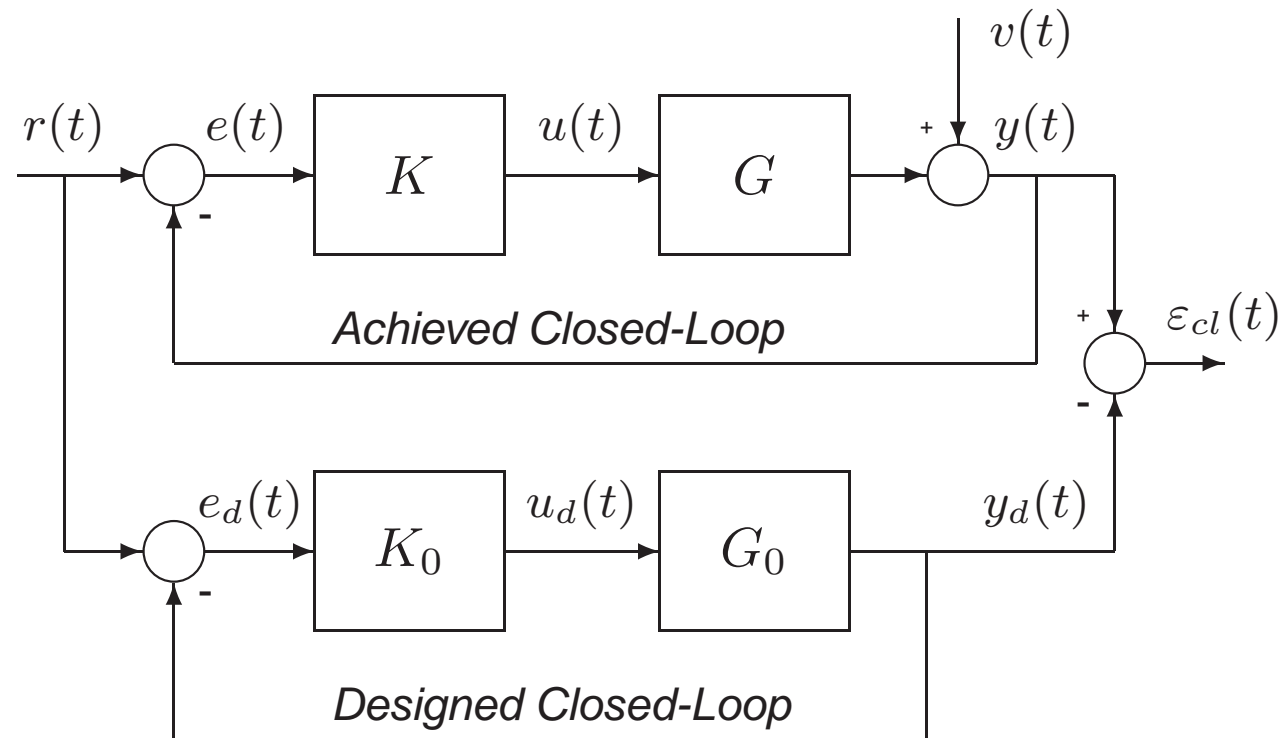
3. Iterative Controller Tuning by Minimizing a Frequency Criterion

- Simple relay tests for gain and phase margins measurement
- Model-free gradient estimation using Bode's integrals

4. Concluding remarks

Introduction

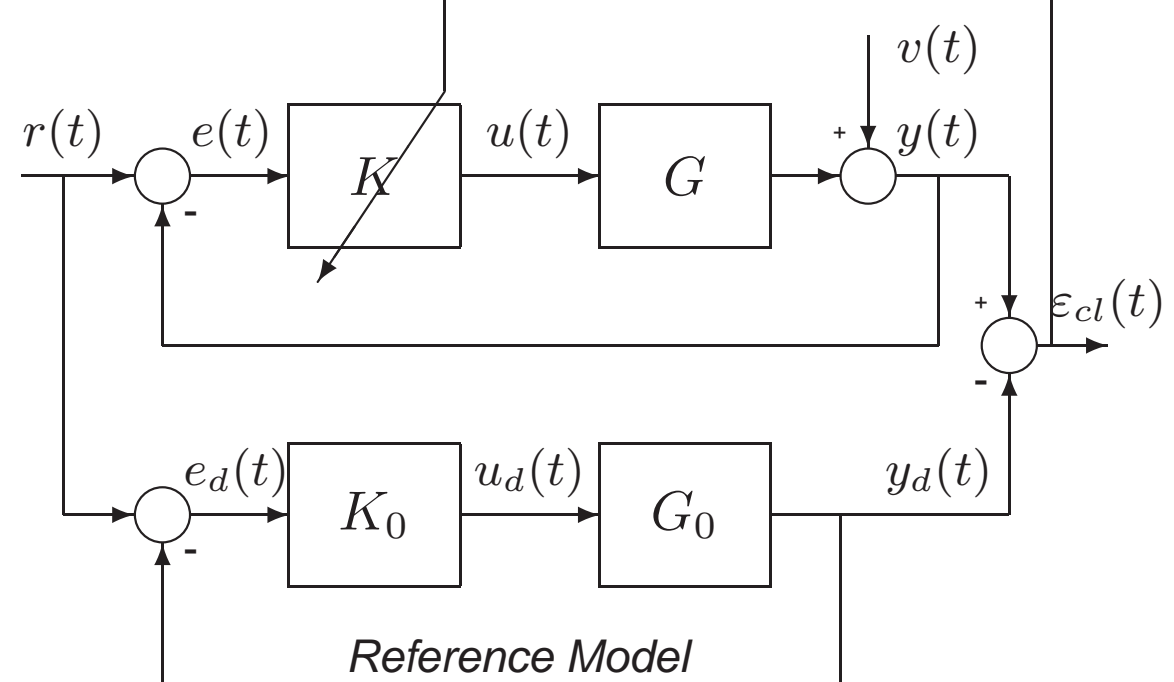
Iterative identification and control



Because of unmodeled dynamics the designed performance cannot be achieved on the real system.

Solution: Closed-loop identification and Controller redesign

- ## 2. Noise effects



Introduction

Gradient estimation

Gradient depends on the *true* closed-loop transfer function

Different approaches:

- Model Reference Adaptive Control (MRAC)[MIT rule, 1958]:

reference model \rightarrow *gradient*

- Self Tuning Regulation (STR)[Astrom, Wittenmark 1973]:

identified model \rightarrow *gradient*

- Iterative Controller Tuning (Adaptation period \gg Sampling period) [Trulsson, Ljung 1985]:

better identified model \rightarrow *better gradient estimate*

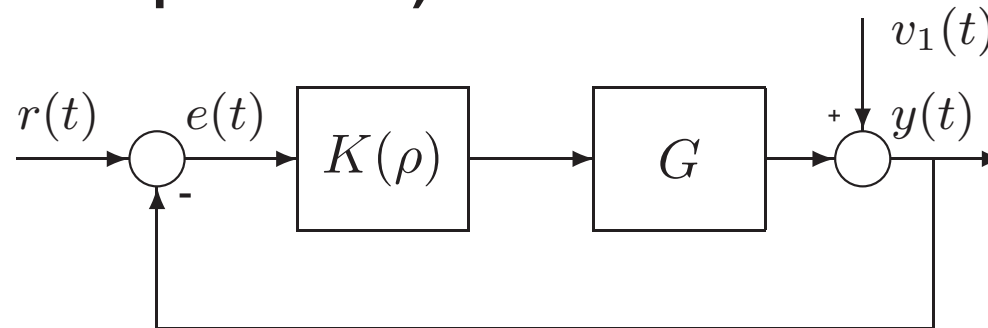
- Iterative Feedback Tuning (IFT) [Hjalmarsson, Gunnarsson, Gevers 1994]:

closed-loop data \rightarrow *gradient (model-free, unbiased)*

Introduction

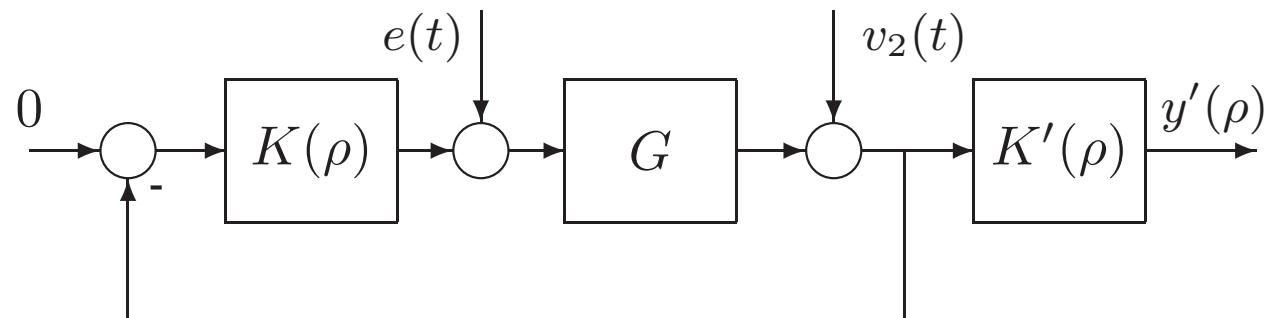
Iterative Feedback Tuning (two experiments)

1. Normal experiment
(criterion evaluation)



$$J(\rho) = \frac{1}{2N} \sum_{t=1}^N E\{[y(\rho, t) - y_d(t)]^2\}$$

2. Gradient experiment
(gradient estimation)



$$J'(\rho) = \frac{1}{N} \sum_{t=1}^N E\{[y(\rho, t) - y_d(t)]y'(\rho, t)\}$$

Unbiased model-free estimation of gradient \rightarrow convergence to a local minimum of the criterion

Introduction

Iterative Feedback Tuning: Properties

- Possibility of minimizing a generalized criterion:

$$J(\rho) = \frac{1}{2N} \sum_{t=1}^N F(q^{-1}) E\{[y(\rho, t) - y_d(t)]^2 + \lambda u^2(\rho, t)\}$$

- Application to non-linear and MIMO systems
- Tuned controller depends on the reference signal and noise characteristics

$$\begin{aligned} \rho^* &= \arg \min_{\rho} \frac{1}{2N} \sum_{t=1}^N E\{[y(\rho, t) - y_d(t)]^2\} \\ &= \arg \min_{\rho} \int_{-\pi}^{\pi} [|T(e^{-j\omega}, \rho) - T_0(e^{-j\omega})|^2 \Phi_r(\omega) + |S(e^{-j\omega}, \rho)|^2 \Phi_v(\omega)] d\omega \end{aligned}$$

$$S(\rho) = \frac{1}{1 + KG} \quad , \quad T(\rho) = \frac{KG}{1 + KG} \quad , \quad T_0 = \frac{K_0 G_0}{1 + K_0 G_0}$$

Iterative Correlation-Based Controller Tuning

- Correlation Approach
- Convergence and Consistency
- Simulation Results
- Reduced-Order Controller Tuning
- Frequency Analysis
- Application to a Magnetic Suspension System

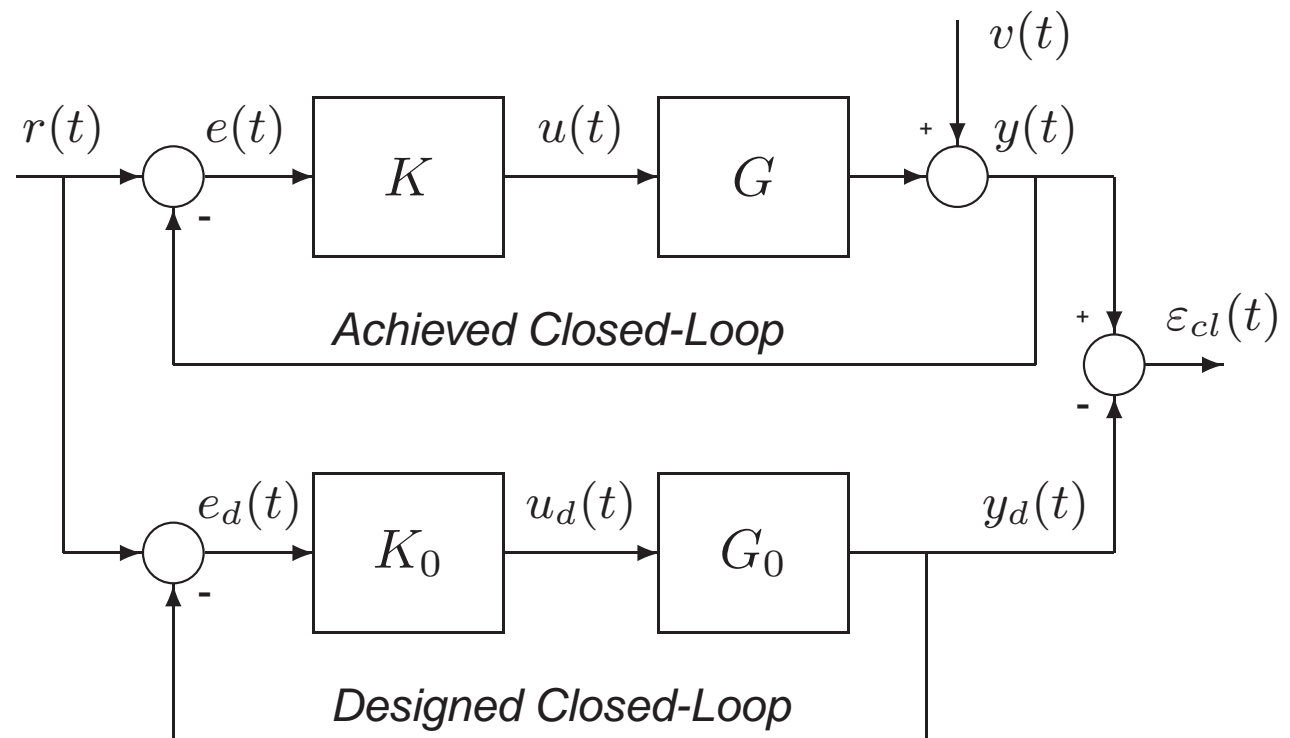
Correlation Approach

Main idea

If $K = K_0$ then

$\varepsilon_{cl}(t)$ contains:

1. filtered noise $v(t)$
2. filtered unmodeled dynamics $(G - G_0)$



Control objective: Find K that makes $\varepsilon_{cl}(t)$ *uncorrelated* with $r(t)$

New controller should compensate for the unmodeled dynamics

Correlation Approach

Correlation Equation:

$$f(\rho) = E\{\zeta(t)\varepsilon_{cl}(\rho, t)\} = 0$$

ρ : Vector of controller parameters

$\varepsilon_{cl}(\rho, t)$: Closed-loop output error

$\zeta(t)$: Vector of instrumental variables, correlated with $r(t)$ and independent of noise $v(t)$

Iterative Solution to the Correlation Equation

$$\rho_{i+1} = \rho_i - \gamma_i [Q_N(\rho_i)]^{-1} \hat{f}(\rho_i)$$

where:

$$\hat{f}(\rho) = \frac{1}{N} \sum_{t=1}^N \zeta(\rho, t) \varepsilon_{cl}(\rho, t)$$

N : number of data

γ_i : scalar positive step size

$Q_N(\rho_i)$: a positive definite matrix

No gradient, no additional experiment and no model is required !!!

Correlation Approach

Newton-Raphson method: can be used in order to improve the convergence speed

$$\begin{aligned} Q_N(\rho_i) = \left. \frac{\partial \hat{f}}{\partial \rho} \right|_{\rho_i} &= \frac{1}{N} \sum_{t=1}^N \left\{ \left. \frac{\partial \zeta(\rho, t)}{\partial \rho} \right|_{\rho_i} \varepsilon_{cl}(\rho_i, t) + \zeta(\rho_i, t) \left. \frac{\partial \varepsilon_{cl}(\rho, t)}{\partial \rho} \right|_{\rho_i} \right\} \\ &\approx \frac{1}{N} \sum_{t=1}^N \zeta(\rho_i, t) \left. \frac{\partial \varepsilon_{cl}(\rho, t)}{\partial \rho} \right|_{\rho_i} \end{aligned}$$

Choice of Instrumental Variables:

Let define:

$$\psi^T(\rho, t) = \frac{\partial \widehat{\varepsilon}_{cl}(\rho, t)}{\partial \rho} \qquad \zeta(\rho, t) = \hat{\psi}(\rho, t)$$

$\psi^T(\rho, t)$: gradient estimate based on an identified model of the plant and real data

$\hat{\psi}(\rho, t)$: noise-free part of the gradient estimate, based on the identified model and simulated data

Convergence and Consistency

Assumptions

A1) The system to be controlled is SISO, LTI, finite order and strictly causal

A2) The reference signal $r(t)$ is persistently exciting of sufficiently high order and uncorrelated with zero-mean finite power disturbance signal $v(t)$

A3) The controller Computed at each iteration stabilizes the closed-loop system

A4) The solution ρ^* exists and is unique:

$$K^* = K_0 \frac{G_0}{G}$$

- G_0 contains the unstable zeros of G
- the order of the estimated controller is large enough to compensate the unmodeled dynamics

Convergence and Consistency

The Controller parameters ρ_i , when $N \rightarrow \infty$ and $i \rightarrow \infty$, converge to the solution of the correlation equation with probability one if:

$$Q(\rho_i) = E\{\zeta(\rho_i, t)\hat{\psi}^T(\rho_i, t)\} \text{ exists and is nonsingular (w.p.1)}$$

Nonsingularity of $Q(\rho_i)$

Theorem: Consider the instrumental variables based on the identified model $\zeta(\rho_i, t)$ and $H(z^{-1})$ as follows:

$$H(z^{-1}) = \frac{\hat{A}(z^{-1})}{\hat{P}(z^{-1})} \frac{P(z^{-1})}{A(z^{-1})}$$

where $A(z^{-1})$ and $P(z^{-1})$ are denominator of the plant and the closed loop system respectively, and $\hat{A}(z^{-1})$ and $\hat{P}(z^{-1})$ are identified ones used in constructions of the IV.

(a) If $r(t)$ is persistently exciting of order ρ (or more) and $H(z^{-1})$ is SPR then Q is nonsingular.

(b) If $r(t)$ is a deterministic periodic signal with period ρ and persistently exciting of order ρ and $H(z^{-1})$ has no pole on the unit circle, then Q is nonsingular.

Convergence and Consistency

Simulation Example (effect of modeling error)

True system:

$$y(t) = \frac{q^{-1} + 0.5q^{-2}}{1 - 1.5q^{-1} + 0.7q^{-2}} u(t)$$

Reference model:

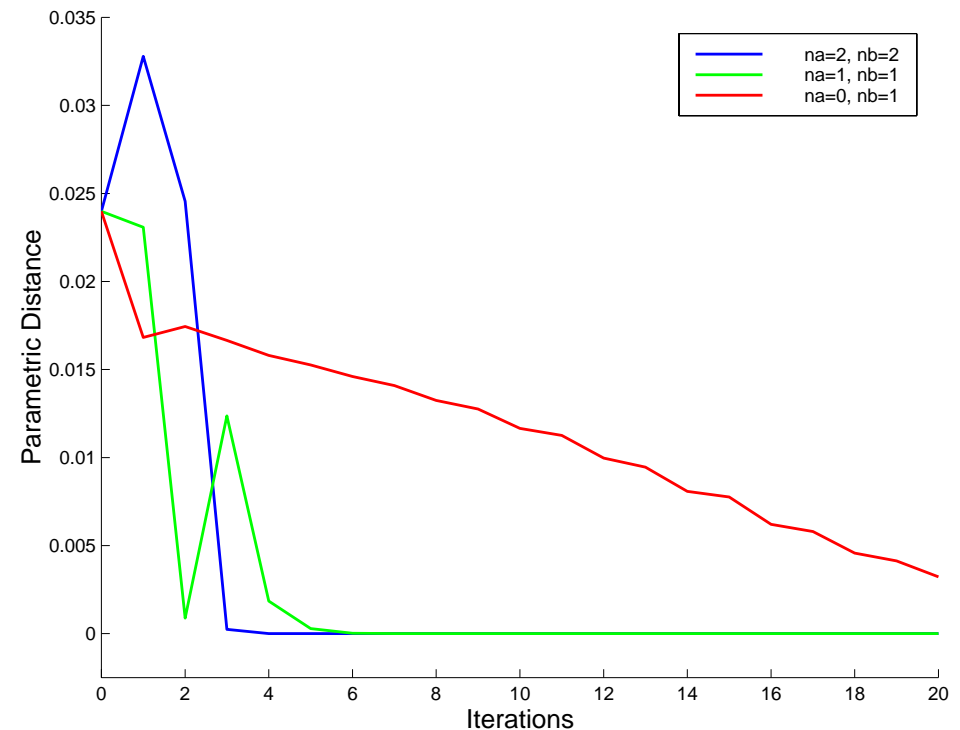
$$\frac{B_m}{A_m} = \frac{-10^{-3}[78q^{-1} + 63q^{-2} + 12q^{-3}]}{1 - 1.578q^{-1} + 0.638q^{-2} - 0.012q^{-3}}$$

The optimal controller:

$$R^* = 1 \text{ and } S^* = -0.0781 - 0.0234q^{-1}$$

The initial controller:

$$R_0 = 1 \text{ and } S_0 = 0.075 + 0.0q^{-1}$$



Modeling error does not affect the convergence of algorithm

Convergence and Consistency

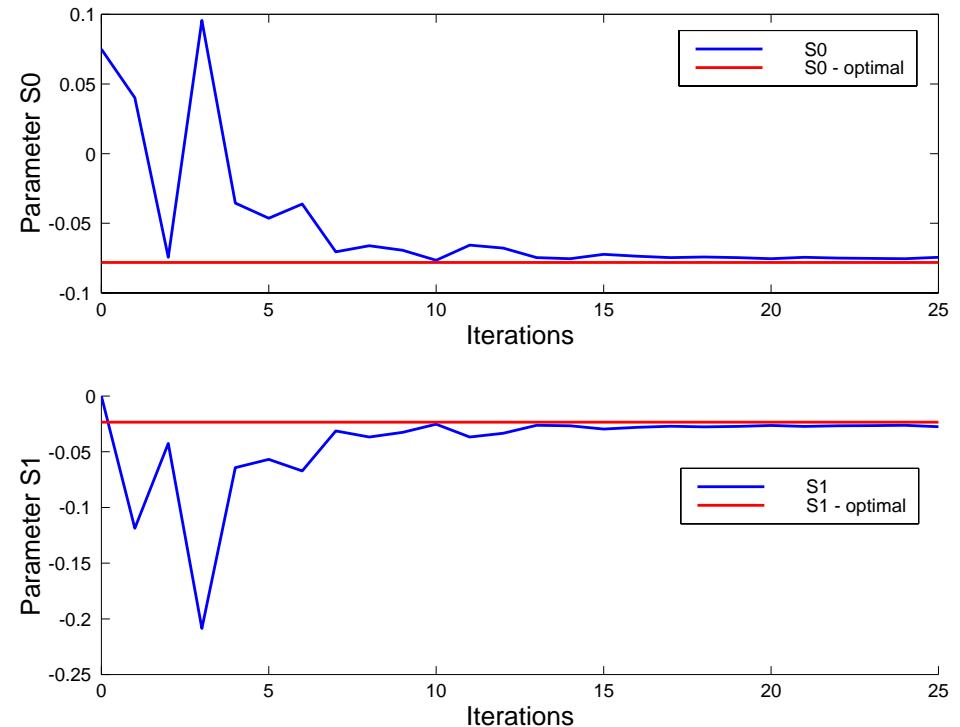
Simulation Example (effect of noise)

True system:

$$y(t) = \frac{q^{-1} + 0.5q^{-2}}{1 - 1.5q^{-1} + 0.7q^{-2}}u(t) + v(t)$$

$$v(t) = \frac{1 + 0.5q^{-1} + 0.5q^{-2}}{1 - 1.5q^{-1} + 0.7q^{-2}}e(t)$$

- Monte-Carlo simulation with 100 runs
- Noise/Signal ratio of 7.5% in terms of variance
- 11-bit PRBS of length 2047 as reference signal
- 25 iterations per simulation
- Plant model is identified with $n_{\hat{A}} = n_{\hat{B}} = 1$



Noise does not affect the convergence of algorithm

Reduced-Order Controller Tuning

In this case there is no solution to the correlation equation

A new criterion is defined as follows:

$$\rho^* = \arg \min_{\rho} J(\rho) = \|f(\rho)\|_2^2 = f^T(\rho)f(\rho)$$

Iterative Solution:

$$\rho_{i+1} = \rho_i - \gamma_i [Q(\rho_i)]^{-1} J'(\rho_i)$$

γ_i : Step size $Q(\rho_i)$: A positive definite matrix $J'(\rho_i)$: Gradient of the criterion

This algorithm converges to a local minimum of the criterion provided that (Robbins-Monro):

- $\sum_{i=0}^{\infty} \gamma_i = \infty$ $\sum_{i=0}^{\infty} \gamma_i^2 < \infty$
- $r(t)$ and $v(t)$ are independent stationary stochastic processes.
- $y(t)$ is bounded at each iteration (closed-loop system is stable).

Frequency Domain Analysis

Consider the following instrumental variables:

$$\zeta^T(t) = [r(t + n_z), r(t + n_z - 1), \dots, r(t), r(t - 1), \dots, r(t - n_z)]$$

So the criterion becomes:

$$J(\rho) = f^T(\rho)f(\rho) = \sum_{\tau=-n_z}^{n_z} R_{\varepsilon r}^2(\tau)$$

where: $R_{\varepsilon r}(\tau) = E\{\varepsilon_{cl}(\rho, t)r(t - \tau)\}$

Using the Parseval's relation for the criterion ($n_z \rightarrow \infty$), we have:

$$\rho^* = \arg \min_{\rho} \int_{-\pi}^{\pi} |\Phi_{\varepsilon r}(\omega)|^2 d\omega = \int_{-\pi}^{\pi} |T(e^{-j\omega}, \rho) - T_0(e^{-j\omega})|^2 \Phi_r^2(\omega) d\omega$$

For the methods which minimize the two norm of closed-loop output error (like IFT, STR), we have:

$$\rho^* = \arg \min_{\rho} \int_{-\pi}^{\pi} [|T(e^{-j\omega}, \rho) - T_0(e^{-j\omega})|^2 \Phi_r(\omega) + |S(e^{-j\omega}, \rho)|^2 \Phi_v(\omega)] d\omega$$

Application to a Magnetic Suspension System

Linearized model:

$$G(s) = \frac{0.1}{0.017s + 1} \frac{15750}{s^2 - 1238}$$

Discrete-time model ($f_s = 100Hz$):

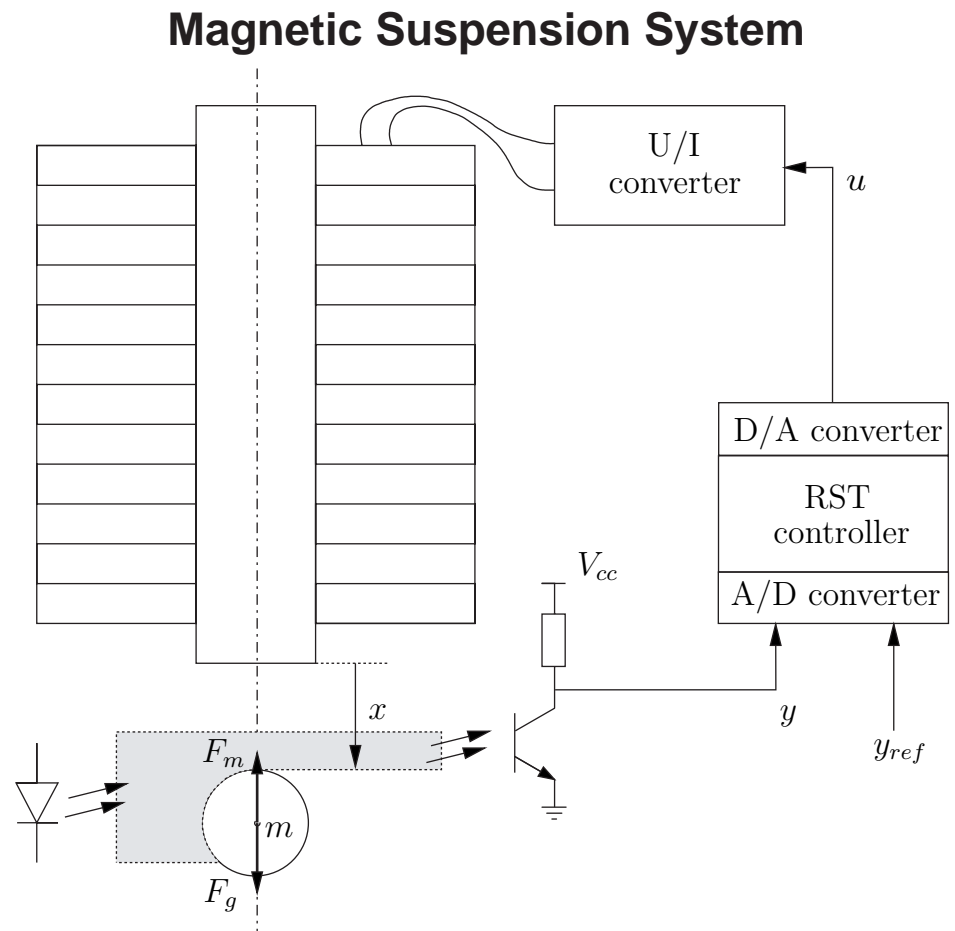
$$G_0(q^{-1}) = \frac{10^{-4}(137q^{-1} + 481q^{-2} + 103q^{-3})}{1 - 2.69q^{-1} + 2.19q^{-2} - 0.56q^{-3}}$$

Initial RST controller:

$$R_0(q^{-1}) = 1 + 0.686q^{-1} + 0.163q^{-2}$$

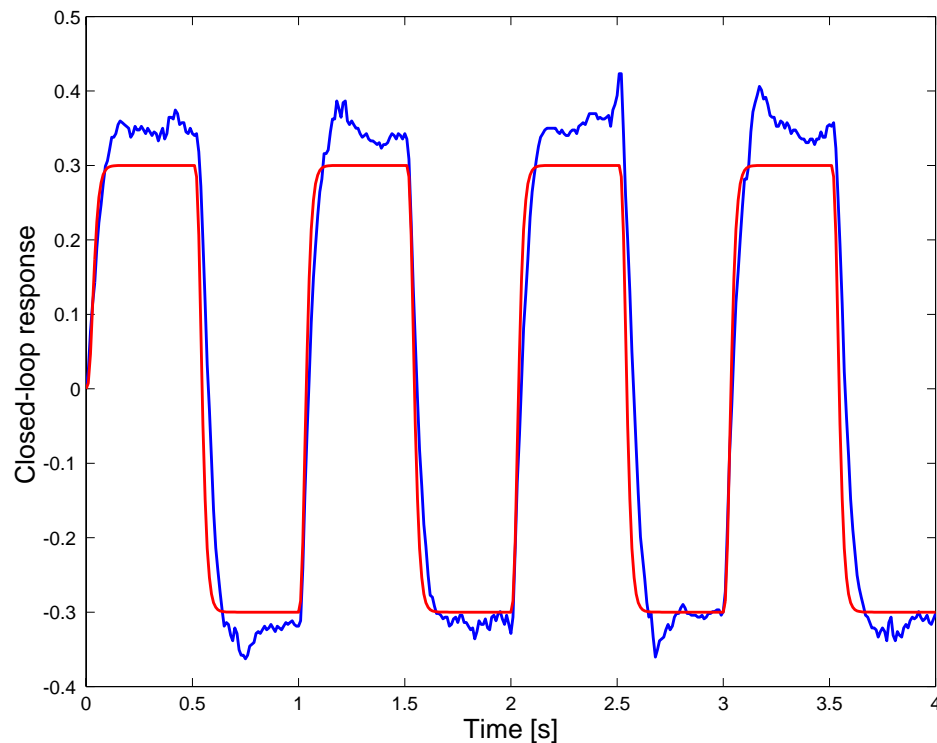
$$S_0(q^{-1}) = 21.86 - 26.77q^{-1} + 8.15q^{-2}$$

$$T_0(q^{-1}) = 1.83$$

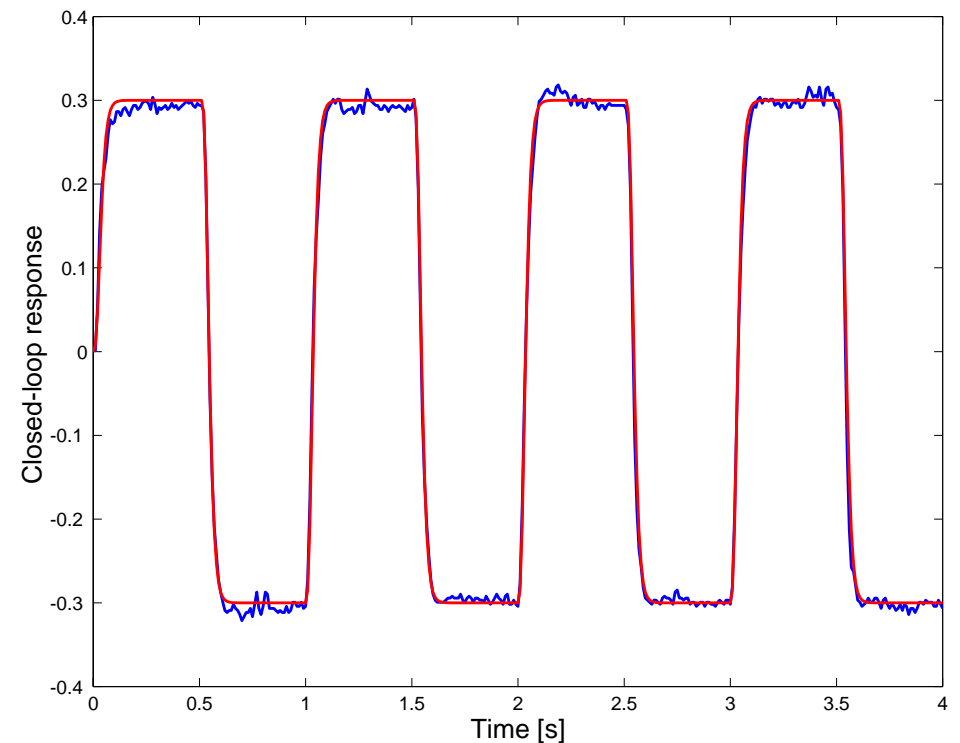


Application to a Magnetic Suspension System

Closed-Loop Response



Initial RST controller and designed response



After 6 iterations using the proposed approach

Iterative Controller Tuning Using Bode's Integrals

- Minimizing a Frequency Criterion in Terms of Phase Margin, Gain Margin and Cross-Over Frequency
- Relay Feedback Tests for Measuring the Robustness Margins
- Using Bode's Integrals for Gradient and Hessian Estimation
- Simulation and Experimental Results

Frequency criteria

The objective is to tune a controller by minimizing iteratively a frequency criterion with the Gauss-Newton method

- **Criteria:**
$$J_1(\rho) = \frac{1}{2} \left[\frac{1}{\omega_d^2} (\omega_c - \omega_d)^2 + \frac{1}{\Phi_d^2} (\Phi_m - \Phi_d)^2 \right]$$
$$J_2(\rho) = \frac{1}{2} \left[\frac{1}{\omega_d^2} (\omega_c - \omega_d)^2 + \frac{1}{\Phi_d^2} (\Phi_m - \Phi_d)^2 + \frac{1}{K_d^2} (K_u - K_d)^2 \right]$$
- **Iterative solution:**
$$\rho_{i+1} = \rho_i - H_{1,2}^{-1} J'_{1,2}(\rho_i)$$

$\omega_{c(d)}$: Measured (desired) crossover frequencies

$\Phi_{m(d)}$: Measured (desired) phase margin

$K_{u(d)}$: Inverse of measured (desired) gain margin

$H_{1,2}$: Hessian of the criterion J_1 or J_2

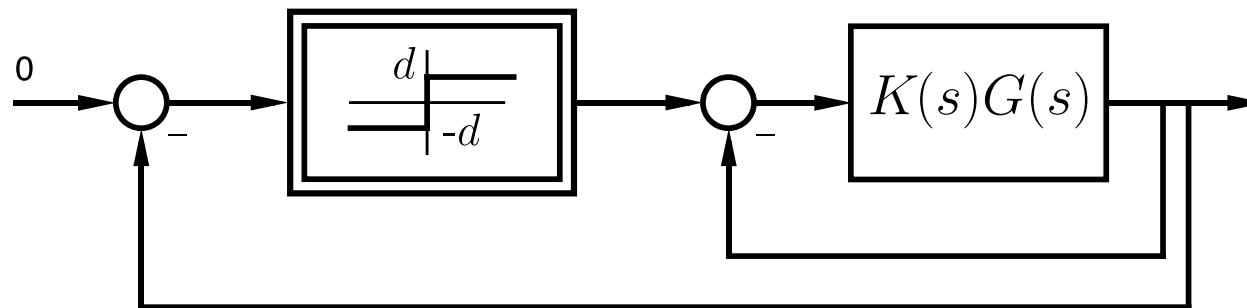
ρ : Vector of the controller parameters

i : iteration number

$J'_{1,2}$: Gradient of the criterion J_1 or J_2

Closed-loop relay test (1)

Gain margin and ultimate frequency ω_u measurement [Aström, Hägglund]



- Relay output is the reference signal of the closed-loop system
- Condition for a limit cycle: $\frac{K(j\omega)G(j\omega)}{1+K(j\omega)G(j\omega)} = -\frac{\pi a}{4d}$
- Identified point: $K(j\omega_u)G(j\omega_u) = -\frac{\pi a}{4d+\pi a} \in (-1, 0)$

Closed-loop relay test (2)

Phase margin and crossover frequency ω_c measurement [Schei, Longchamp, Piguet]

- Condition for a limit cycle:

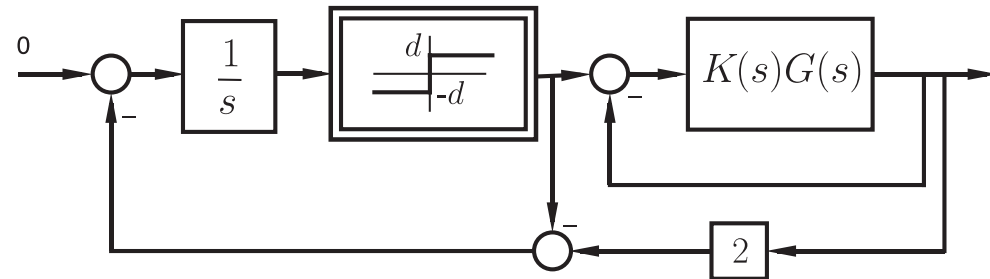
$$\frac{1}{j\omega} \frac{K(j\omega)G(j\omega) - 1}{K(j\omega)G(j\omega) + 1} = -\frac{\pi a}{4d}$$

- Identified point:

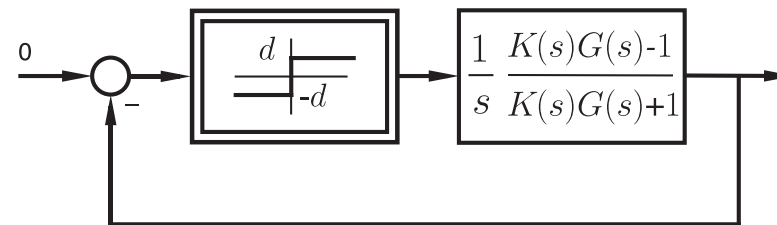
$$K(j\omega_c)G(j\omega_c) = \frac{1 - j \frac{\omega_c \pi a}{4d}}{1 + j \frac{\omega_c \pi a}{4d}}$$

$$|K(j\omega_c)G(j\omega_c)| = 1$$

$$\angle K(j\omega_c)G(j\omega_c) = -2 \arctan\left(\frac{\pi a \omega_c}{4d}\right)$$



\Leftrightarrow



Procedure for phase margin adjustment (1)

- **Criterion:** $J(\rho) = \frac{1}{2} \left[\frac{1}{\omega_d^2} (\omega_c - \omega_d)^2 + \frac{1}{\Phi_d^2} (\Phi_m - \Phi_d)^2 \right]$
- **Iterative solution:** $\rho_{i+1} = \rho_i - H^{-1} J'(\rho_i)$

ρ : Vector of the controller parameters

$\omega_{c(d)}$: Measured (desired) crossover frequencies

i : iteration number

$\Phi_{m(d)}$: Measured (desired) phase margin

- Gradient: $J'(\rho) = \frac{1}{\omega_d^2} (\omega_c - \omega_d) \frac{\partial \omega_c}{\partial \rho} + \frac{1}{\Phi_d^2} (\Phi_m - \Phi_d) \Phi'_m$

- Hessian: $H(\rho) = J''(\rho) = \frac{1}{\omega_d^2} \frac{\partial \omega_c}{\partial \rho} \left(\frac{\partial \omega_c}{\partial \rho} \right)^T + \frac{1}{\Phi_d^2} \Phi'_m (\Phi'_m)^T$
 $+ \frac{1}{\omega_d^2} (\omega_c - \omega_d) \frac{\partial^2 \omega_c}{\partial \rho^2} + \frac{1}{\Phi_d^2} (\Phi_m - \Phi_d) \Phi''_m$

$$\approx \frac{1}{\omega_d^2} \frac{\partial \omega_c}{\partial \rho} \left(\frac{\partial \omega_c}{\partial \rho} \right)^T + \frac{1}{\Phi_d^2} \Phi'_m (\Phi'_m)^T$$

Procedure for phase margin adjustment (2)

Approximation of $\frac{\partial \omega_c}{\partial \rho}$:

$$\frac{d \ln |L(j\omega_c)|}{d\rho} = 0 \quad \Rightarrow \quad \frac{\partial \ln |L(j\omega_c)|}{\partial \rho} + \frac{\partial \ln |L(j\omega)|}{\partial \omega} \bigg|_{\omega_c} \frac{\partial \omega_c}{\partial \rho} = 0$$

$$\Rightarrow \frac{\partial \omega_c}{\partial \rho} = - \frac{\partial \ln |L(j\omega_c)|}{\partial \rho} \left[\frac{\partial \ln |L(j\omega)|}{\partial \omega} \bigg|_{\omega_c} \right]^{-1}$$

Approximation of Φ'_m :

$$\Phi'_m = \frac{\partial \Phi_m}{\partial \rho} + \frac{\partial \Phi_m}{\partial \omega} \bigg|_{\omega_c} \frac{\partial \omega_c}{\partial \rho} = \frac{\partial \angle L(j\omega_c)}{\partial \rho} + \frac{\partial \angle L(j\omega)}{\partial \omega} \bigg|_{\omega_c} \frac{\partial \omega_c}{\partial \rho}$$

$$\frac{\partial \angle L(j\omega)}{\partial \omega} \bigg|_{\omega_c} = \frac{\partial \angle K(j\omega)}{\partial \omega} \bigg|_{\omega_c} + \frac{\partial \angle G(j\omega)}{\partial \omega} \bigg|_{\omega_c}$$

Bode's integrals: Derivative of amplitude

- For a stable minimum-phase transfer function $G(s)$, the phase of the system at any ω_0 is given by:

$$\angle G(j\omega_0) = \frac{1}{\pi} \int_{-\infty}^{+\infty} \frac{d \ln |G(j\omega)|}{d\nu} \ln \coth \frac{|\nu|}{2} d\nu$$

where $\nu = \ln \frac{\omega}{\omega_0}$

- Since: $\ln \coth \frac{|\nu|}{2}$ decreases rapidly as ω deviates from ω_0

The slope of the Bode plot is almost constant in the neighborhood of ω_0

$$\angle G(j\omega_0) \approx \frac{\pi}{2} \left. \frac{d \ln |G(j\omega)|}{d\nu} \right|_{\omega_0}$$

\Rightarrow

$$s_a(\omega_0) = \left. \frac{d \ln |G(j\omega)|}{d\nu} \right|_{\omega_0} \approx \frac{2}{\pi} \angle G(j\omega_0)$$

Bode's integrals: Derivative of phase

- For a stable minimum-phase transfer function $G(s)$, the logarithm of the system amplitude at any ω_0 is given by:

$$\ln |G(j\omega_0)| = \ln |K_g| - \frac{\omega_0}{\pi} \int_{-\infty}^{+\infty} \frac{d(\angle G(j\omega)/\omega)}{d\nu} \ln \coth \frac{|\nu|}{2} d\nu$$

where $\nu = \ln \frac{\omega}{\omega_0}$, and K_g is the static gain of the plant

- In the same way

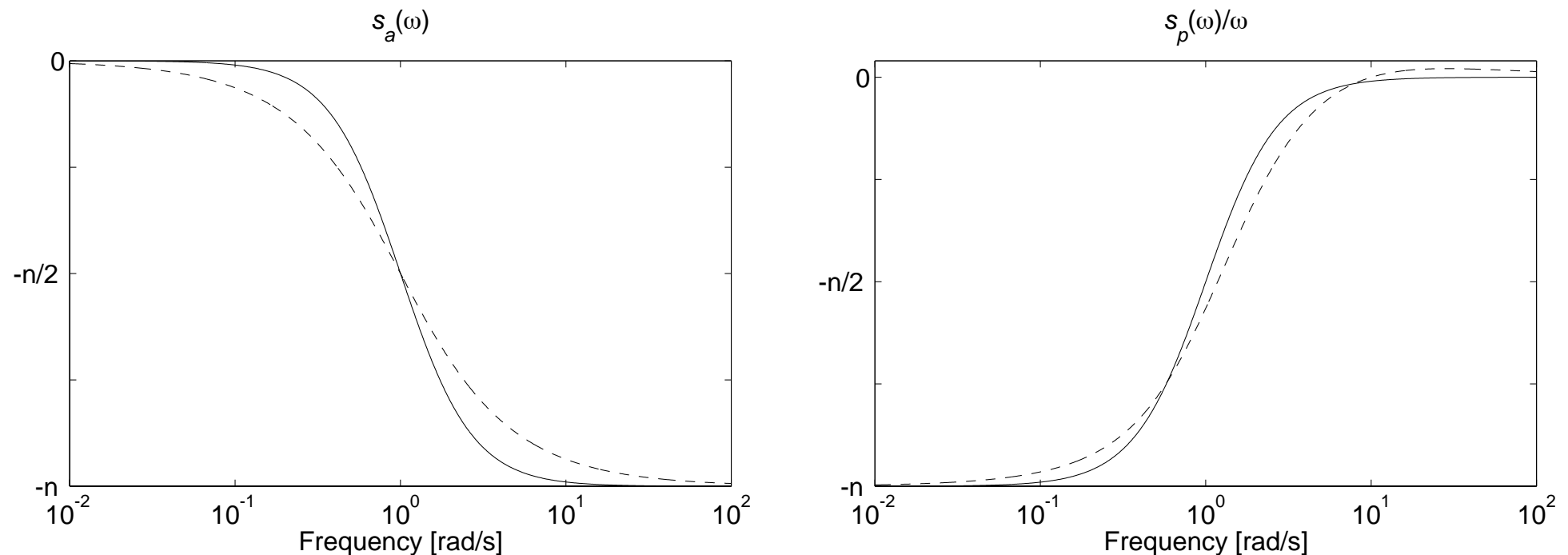
$$\ln |G(j\omega_0)| \approx \ln |K_g| - \frac{\omega_0}{\pi} \left. \frac{d(\angle G(j\omega)/\omega)}{d\nu} \right|_{\omega_0} \frac{\pi^2}{2}$$

$$\Rightarrow \quad s_p(\omega_0) = \omega_0 \left. \frac{d\angle G(j\omega)}{d\omega} \right|_{\omega_0} \approx \angle G(j\omega_0) + \frac{2}{\pi} [\ln |K_g| - \ln |G(j\omega_0)|]$$

Bode's Integrals (Precision of estimates)

Comparison of true $S_a(\omega)$ and $S_p(\omega)/\omega$ and estimates based on Bode's integrals for:

$$G(s) = \frac{1}{(s + 1)^n}$$



True values (solid line) and estimates (dashed line)

Procedure for phase and gain margin adjustment (1)

- **Criterion:** $J(\rho) = \frac{1}{2} \left[\frac{1}{\omega_d^2} (\omega_c - \omega_d)^2 + \frac{1}{\Phi_d^2} (\Phi_m - \Phi_d)^2 + \frac{1}{K_d^2} (K_u - K_d)^2 \right]$
- **Iterative solution:** $\rho_{i+1} = \rho_i - H^{-1} J'(\rho_i)$

$\omega_{c(d)}$: Measured (desired) crossover frequencies

ρ : Vector of the controller parameters

$\Phi_{m(d)}$: Measured (desired) phase margin

i : iteration number

$K_{u(d)}$: Inverse of measured (desired) gain margin

- Gradient: $J'(\rho) = \frac{1}{\omega_d^2} (\omega_c - \omega_d) \frac{\partial \omega_c}{\partial \rho} + \frac{1}{\Phi_d^2} (\Phi_m - \Phi_d) \Phi'_m + \frac{1}{K_d^2} (K_u - K_d) K'_u$

- Hessian: $H(\rho) = J''(\rho)$

$$\approx \frac{1}{\omega_d^2} \frac{\partial \omega_c}{\partial \rho} \left(\frac{\partial \omega_c}{\partial \rho} \right)^T + \frac{1}{\Phi_d^2} \Phi'_m (\Phi'_m)^T + \frac{1}{K_d^2} K'_u (K'_u)^T$$

Procedure for phase and gain margin adjustment (2)

Approximation of K'_u :

$$K'_u = \frac{\partial |L(j\omega_u)|}{\partial \rho} + \frac{\partial |L(j\omega)|}{\partial \omega} \Big|_{\omega_u} \frac{\partial \omega_u}{\partial \rho}$$

$$\frac{\partial |L(j\omega)|}{\partial \omega} \Big|_{\omega_u} = |L(j\omega_u)| \frac{\partial \ln |L(j\omega)|}{\partial \omega} \Big|_{\omega_u} \approx K_u \frac{2\angle L(j\omega_u)}{\pi \omega_u} = -\frac{2K_u}{\omega_u}$$

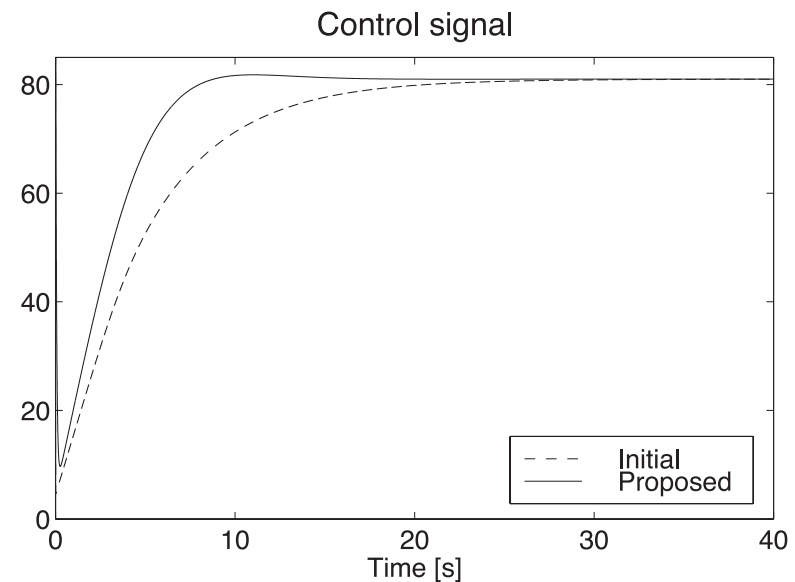
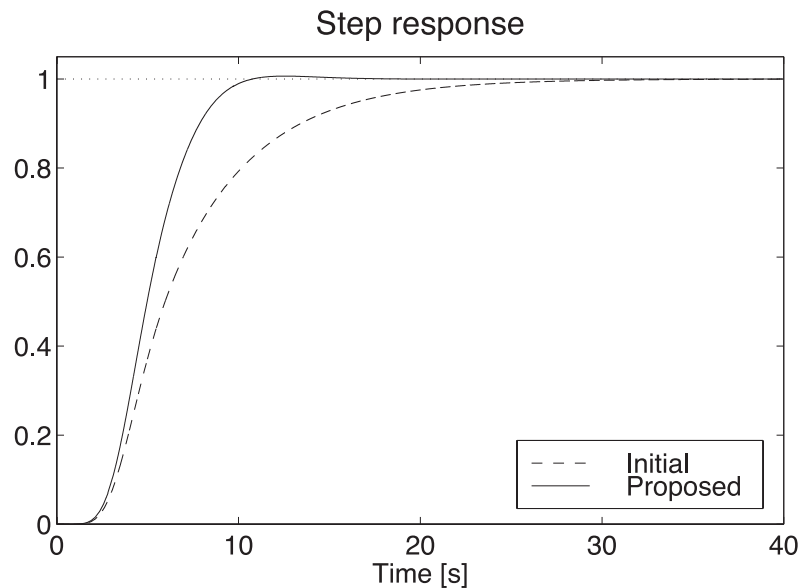
Approximation of $\frac{\partial \omega_u}{\partial \rho}$:

$$\frac{\partial \angle L(j\omega_u)}{\partial \rho} + \frac{\partial \angle L(j\omega)}{\partial \omega} \Big|_{\omega_u} \frac{\partial \omega_u}{\partial \rho} = 0 \quad \Rightarrow \quad \frac{\partial \omega_u}{\partial \rho} = -\frac{\partial \angle L(j\omega_u)}{\partial \rho} \left(\frac{\partial \angle L(j\omega)}{\partial \omega} \Big|_{\omega_u} \right)^{-1}$$

$$\frac{\partial \angle L(j\omega)}{\partial \omega} \Big|_{\omega_u} = \frac{\partial \angle K(j\omega)}{\partial \omega} \Big|_{\omega_u} + \frac{\partial \angle G(j\omega)}{\partial \omega} \Big|_{\omega_u} \approx \frac{\partial \angle K(j\omega)}{\partial \omega} \Big|_{\omega_u} + \frac{s_p(\omega_u)}{\omega_u}$$

Simulation results

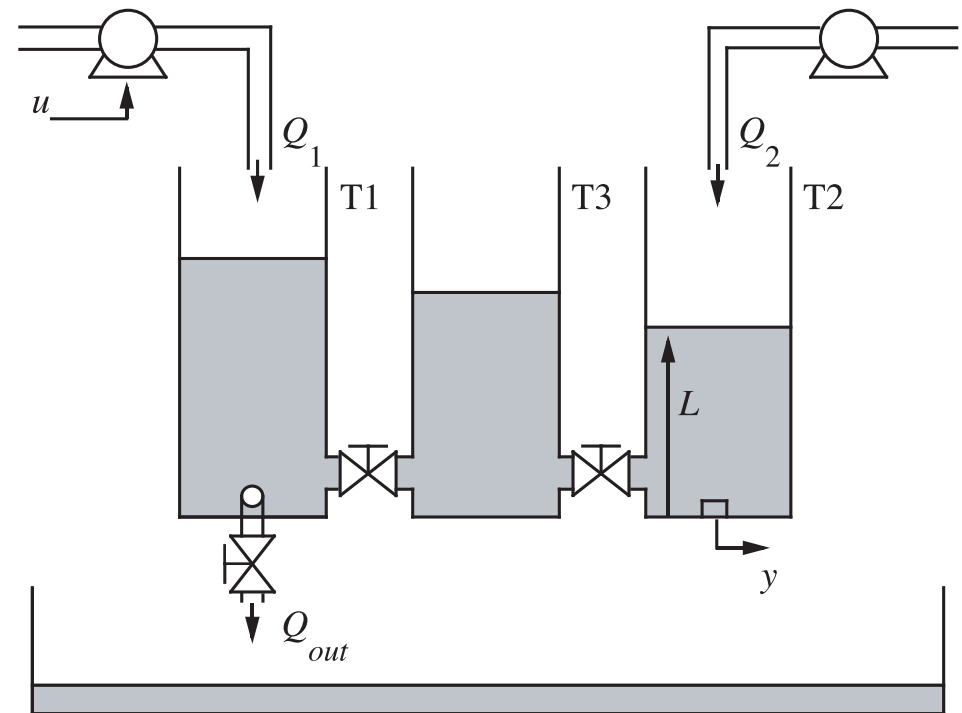
- Model: $G(s) = \frac{e^{-0.3s}}{(s^2+2s+3)^3(s+3)}$
- Initial Controller: $K(s) = 4.5 \left(1 + \frac{1}{0.41s} + 0.033s\right)$ (κ - τ tuning rule)
- Measured performances: $\Phi_m = 78.5^\circ, \omega_c = 0.139 \text{ rad/s}, K_u = \frac{1}{4.39}$
- Specifications: $\Phi_d = 70^\circ, \omega_d = 0.2 \text{ rad/s}, K_d = \frac{1}{3}$
- New controller (1 iteration): $K(s) = 4.93 \left(1 + \frac{1}{0.316s} + 0.125s\right)$
- Obtained performance: $\Phi_m = 66^\circ, \omega_c = 0.199 \text{ rad/s}, K_m = \frac{1}{2.97}$



Experimental Results (1)

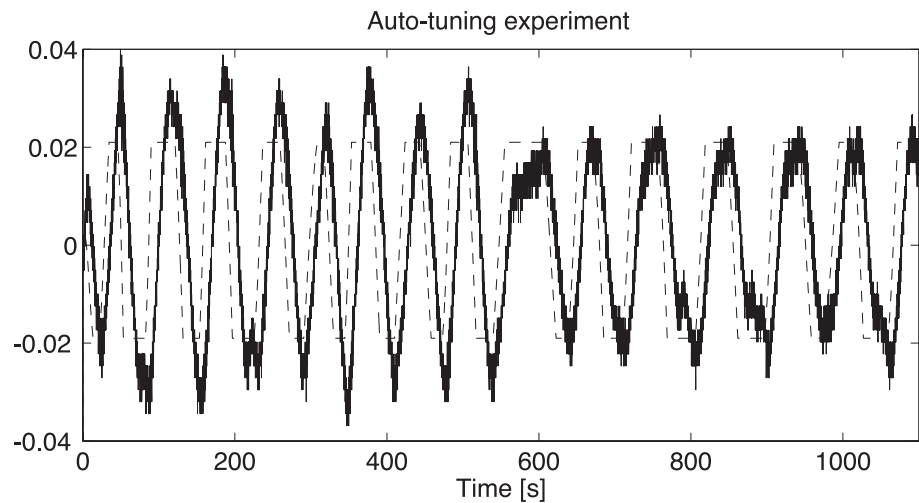
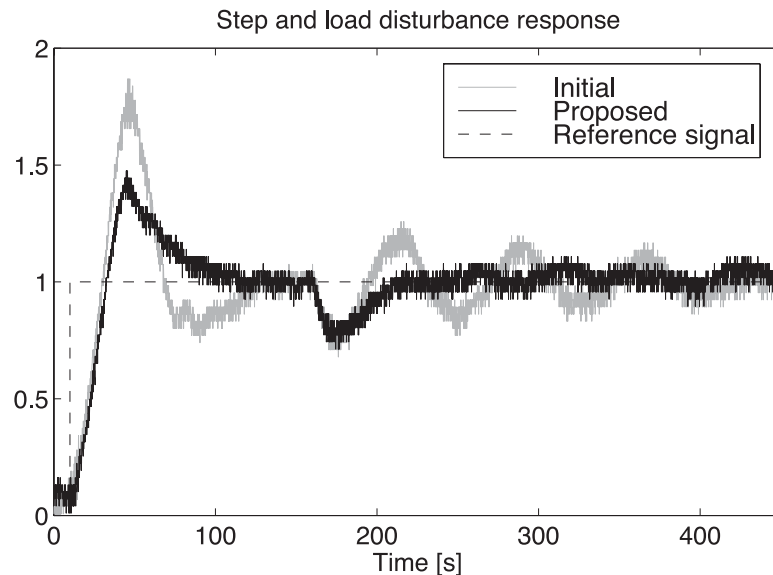
Three-tank system

- T1, T2 and T3: interconnected cylinders
- L : water level of T2
- Q_1 , Q_2 and Q_{out} : flow rates
- Input u : controls the flow rate Q_1
- Output y : proportional to the level L
- Disturbance: flow rate Q_2



Experimental Results (2)

- Initial Controller: $K(s) = 29.3 \left(1 + \frac{1}{20.84s} + 4.72s\right)$ (κ - τ tuning rule)
- Measured performances: $\Phi_m = 64^\circ$, $\omega_c = 0.097$ rad/s
- Specifications: $\Phi_d = 80^\circ$, $\omega_d = 0.08$ rad/s, $T_i = 4T_d$
- New controller (1 iteration): $K(s) = 20.4 \left(1 + \frac{1}{31.5s} + 7.88s\right)$
- Obtained performance: $\Phi_m = 86.2^\circ$, $\omega_c = 0.085$ rad/s



Concluding Remarks

- Experiment-based tuning methods are appropriate for restricted complexity controller design. No model is required or the model is not directly involved in controller tuning.
- Two new approaches for iterative controller tuning are proposed:

Correlation approach

- Making the closed-loop output error *uncorrelated* with the reference signal, can be used as an *objective* for controller tuning.
- Parametric convergence of the controller is not affected by noise and modeling errors.

Frequency approach

- Takes advantage of the Bode's integrals to estimate the gradient of a frequency criterion
- Converges in a few iterations to the minimum of the frequency criterion
- No parametric model is required

References

1. E. Trulsson and L. Ljung. Adaptive control based on explicit criterion minimization. *Automatica*, 21(4):385–399, 1985.
2. H. Hjalmarsson, M. Gevers, S. Gunnarsson, and O. Lequin. Iterative feedback tuning: Theory and application. *IEEE Control Systems Magazine*, pages 26–41, 1998.
3. H. Hjalmarsson. Iterative feedback tuning – an overview. *International Journal of Adaptive Control and Signal Processing*, 16:373–395, 2002.
4. A. Karimi, L. Mišković, and D. Bonvin. Convergence analysis of an iterative correlation-based controller tuning method. In *15th IFAC World Congress, Barcelona, Spain*, July 2002.
5. A. Karimi, L. Mišković, and D. Bonvin. Iterative correlation-based controller tuning: Application to a magnetic suspension system. *Control Engineering Practice*, to appear, 2002.
6. A. Karimi, L. Mišković, and D. Bonvin. Iterative correlation-based controller tuning: Frequency-domain analysis. In *41st IEEE-CDC, Las Vegas, USA*, December 2002.
7. A. Karimi, D. Garcia, and R. Longchamp. Iterative controller tuning using Bode’s integrals. In *41st IEEE-CDC, Las Vegas, USA*, December 2002.