

# Développement et validation d'une métasurface acoustique orientable large bande composée de résonateurs électroacoustiques actifs

Hervé Lissek, Etienne Rivet, Thomas Laurence, Romain Fleury

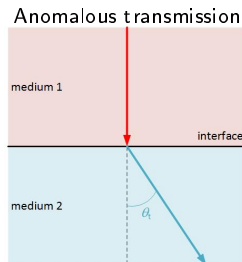
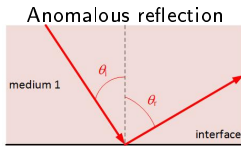
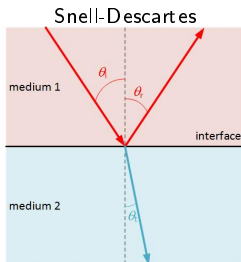
Ecole Polytechnique Fédérale de Lausanne, Switzerland

April 24, 2017

# Acoustic metasurfaces

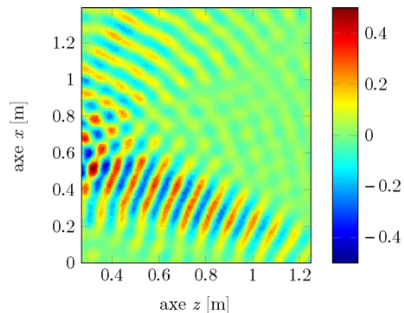
## Principle :

Interfaces breaking the Snell-Descartes laws of refraction.

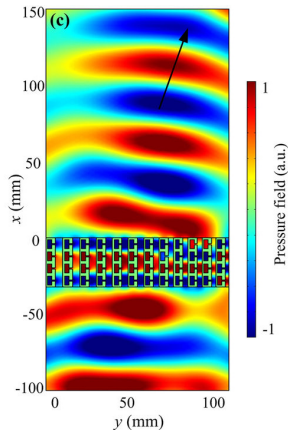


# Acoustic metasurfaces

State of the art :



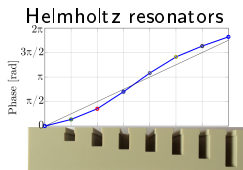
C. Faure et al, *Applied Physics Letters* **108**, 064103 (2016)



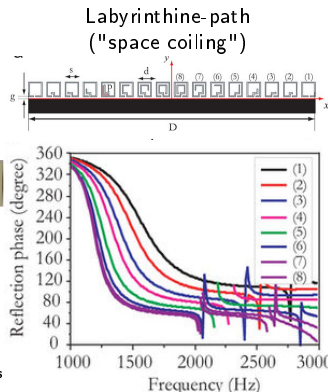
J. Lan et al, *Scientific Reports* **7**, 10587 (2017)

# Acoustic metasurfaces

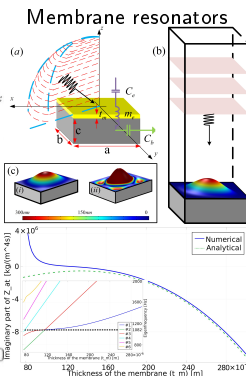
## State of the art : Realizations



C. Faure et al, Applied Physics Letters 108, 064103 (2016)



K. Song et al, Scientific Reports 94, 014302(2016)

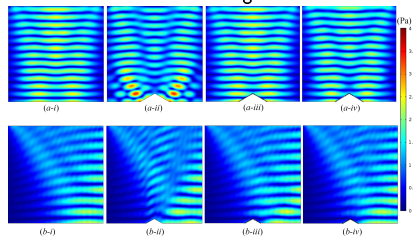


H. Esfahani et al, Physical Review B 6, 32300 (2016)

# Acoustic metasurfaces

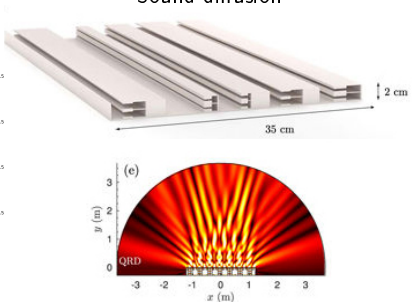
## State of the art : Applications

### Cloaking



H. Esfahlani et al, Physical Review B 6, 32300 (2016)

### Sound diffusion



N. Jimenez et al, Scientific Reports 7, 5389 (2017)

# Motivations

- **Reconfigurability**  
Passive designs only allow for fixed reflection (resp. transmission) characteristics.  
No possibility to reconfigure them on-the-fly.
- **Broadband properties**  
The reported (passive) concepts rely on resonant behaviours (labyrinthine, Helmholtz resonators, membranes-based resonators, etc.).  
The achieved properties only hold around a prescribed frequency, with narrow-band efficiency.
- **Lossless reflection (resp. transmission)**  
Acoustic resonators generally yield a certain amount of losses, that lower the reflection (resp. transmission) efficiency.  
Difficulties to ensure total reflection (resp. transmission) on passive metasurface.

→ active concepts

## Metasurface principle

### Proposed geometry of Acoustic Metasurface (reflection)

2D arrangement of  $M \times N$  small vibrating circular pistons of same radius  $r_d$ .

Subwavelength condition :  $2r_d < \frac{\lambda}{10}$ , where  $\lambda = \frac{c_0}{2\pi f}$  is the wavelength and

$c_0 = 343\text{m}\cdot\text{s}^{-1}$  the sound celerity in the air.

→ maximum radius of 34 mm up to 500 Hz.

Each piston presents an individual reflection coefficient  $\Gamma_{m,n}(f) = A^{mn}(f)e^{j\Psi^{mn}(f)}$ .  
Assuming  $A^{mn}(f)$  is the same  $\forall(m, n)$

### Anomalous reflection condition :

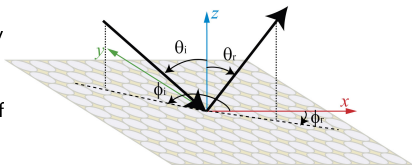
Incident harmonic plane wave (frequency  $f$ )

Incident angle  $(\theta_i, \phi_i)$ .

→ Anomalous reflected angle  $(\theta_r, \phi_r)$  if  
(Generalized Snell-Descartes law) :

$$\Psi^{mn} = \psi_0 - \frac{2\pi f}{c_0}(2r_d) [m(\sin \theta_r \cos \phi_r + \sin \theta_i \cos \phi_i) - n(\sin \theta_r \sin \phi_r + \sin \theta_i \sin \phi_i)]$$

$\psi_0$  : phase reference over the metasurface (eg. central cell)



## Membrane resonator unit-cell

Unit-cell : loudspeaker diaphragm (SDOF mechanical resonator)

Acoustic specific impedance :

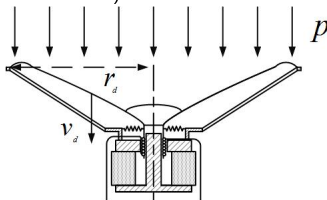
$$Z_{as}(\omega) = \frac{1}{S_d} \left( j\omega M_{ms} + R_{ms} + \frac{1}{j\omega C_{ms}} \right)$$

$M_{ms}$  : moving mass,

$R_{ms}$  : mechanical resistance,

$C_{ms}$  : mechanical compliance,

$S_d$  : diaphragm area



Unit-cell size  $\ll \lambda$

→ reflection coefficient independent on  $(\theta_i, \phi_i)$  :

$$\begin{aligned} \Gamma(\omega) &= \frac{(R_{ms} - S_d Z_c) + j \left( \omega M_{ms} - \frac{1}{\omega C_{ms}} \right)}{(R_{ms} + S_d Z_c) + j \left( \omega M_{ms} - \frac{1}{\omega C_{ms}} \right)} \\ &= \frac{-\left(\frac{f}{f_s}\right)^2 + j \left(\frac{f}{f_s}\right) \frac{1}{Q_s} \left(1 - \frac{1}{r_s}\right) + 1}{-\left(\frac{f}{f_s}\right)^2 + j \left(\frac{f}{f_s}\right) \frac{1}{Q_s} \left(1 + \frac{1}{r_s}\right) + 1} \end{aligned}$$

$$Z_c = \rho_0 c_0 \text{ with } \rho_0 = 1.2 \text{ kg} \cdot \text{m}^{-3}$$

$$r_s = \frac{R_{ms}}{S_d Z_c}$$

$$f_s = \frac{1}{2\pi \sqrt{M_{ms} C_{ms}}}$$

$$Q_s = \frac{1}{R_{ms}} \sqrt{\frac{M_{ms}}{C_{ms}}}$$



## Membrane resonator unit-cell

Unit-cell : loudspeaker diaphragm (SDOF mechanical resonator)

Acoustic specific impedance :

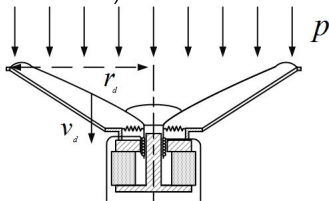
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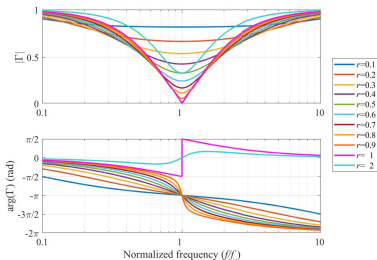


Unit-cell size  $\ll \lambda$

→ reflection coefficient independent on  $(\theta_i, \phi_i)$  :

$$\begin{aligned} \Gamma(\omega) &= \frac{(R_{ms} - S_d Z_c) + j \left( \omega M_{ms} - \frac{1}{\omega C_{ms}} \right)}{(R_{ms} + S_d Z_c) + j \left( \omega M_{ms} - \frac{1}{\omega C_{ms}} \right)} \\ &= \frac{-\left(\frac{f}{f_s}\right)^2 + j \left(\frac{f}{f_s}\right) \frac{1}{Q_s} \left(1 - \frac{1}{r_s}\right) + 1}{-\left(\frac{f}{f_s}\right)^2 + j \left(\frac{f}{f_s}\right) \frac{1}{Q_s} \left(1 + \frac{1}{r_s}\right) + 1} \end{aligned}$$

Fixed  $Q_s = 1$



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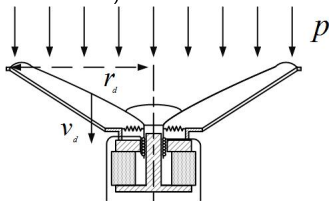
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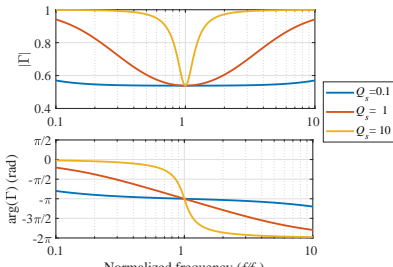


Unit-cell size  $\ll \lambda$

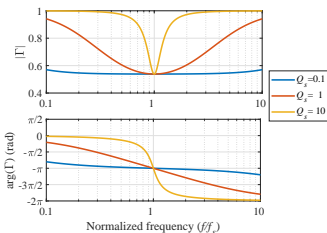
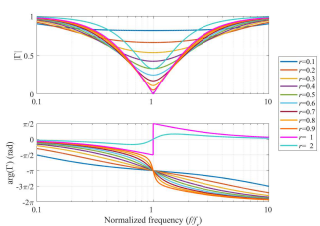
→ reflection coefficient independent on  $(\theta_i, \phi_i)$  :

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Fixed  $r_s = 1/3$



## (passive) design guidelines



In order to ensure desired reflection properties still hold over a significant frequency bandwidth around the natural frequency of the loudspeaker, we need to set the resistance and quality factor of the (passive) loudspeaker diaphragm.

Selected loudspeaker : MONACOR SPX-30M

Parameter	Symbol	Value	Unit
Effective piston area	$S_d$	32	$\text{cm}^2$
Effective piston radius	$r_d$	32	cm
Mechanical mass	$M_{ms}$	3.17	g
Mechanical resistance	$R_{ms}$	0.75	$\text{N.s.m}^{-1}$
Mechanical compliance (with enclosure)	$C_{mc}$	$184.10^{-6}$	$\text{m.N}^{-1}$
Force factor	$B\ell$	3.67	N/A
Resonance frequency	$f_s$	208	Hz
Loss factor	$r_s$	0.57	
Quality factor	$Q_s$	5.5	

It yields :

$$r_s \in [0.3 - 0.6] \rightarrow R_{ms} \approx \frac{S_d Z_c}{2}$$

$$Q_s \in [1 - 10]$$

## Metasurface design strategy

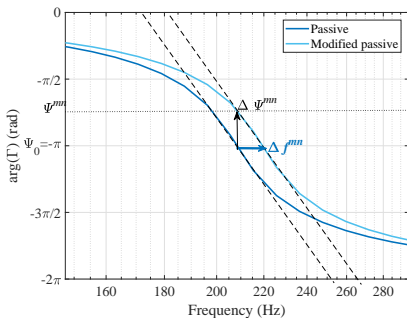
Question : how to impose desired reflection phases  $\Psi^{mn}$  at resonance frequency  $f_s$  ?

## Metasurface design strategy

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Reflection phase linearly decreases with frequency, turning (almost) from 0 to  $-2\pi$  over a given frequency band (at least one octave around  $f_s$ ).

If  $Q_s$  and  $r_s$  are preserved on all unit-cells, a simple resonance shift  $\Delta f^{mn}$  allows assigning prescribed reflection phases  $\Psi^{mn}$ .



Target acoustic impedance :

$$Z_{at}^{mn}(\omega) = \frac{1}{S_d} \left( j\omega \mu_M^{mn} M_{ms} + \mu_R R_{ms} + \frac{1}{j\omega \mu_C^{mn} C_{ms}} \right)$$

with

- $Q_s = 6$

- $\mu_R = r_s = \frac{1}{3}$

And the control parameters :

- $\frac{f_s + \Delta f^{mn}}{f_s} = \frac{1}{\sqrt{\mu_M^{mn} \mu_C^{mn}}}$

- $\mu_C^{mn} = \frac{1}{2\pi Z_c S_d C_{ms} Q_s} \frac{1}{(f_s + \Delta f^{mn}) \mu_R}$

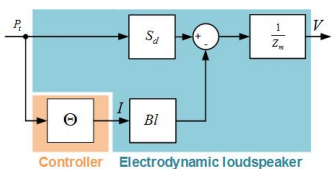
- $\mu_M^{mn} = \frac{Z_c Q_s S_d}{2\pi C_{ms}} \frac{\mu_R}{(f_s + \Delta f^{mn})}$

# Electroacoustic resonator

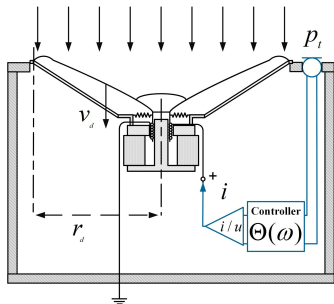
## Electroacoustic resonator principle

Specific feedforward control :

Microphone-based feedforward control,  
through voltage-controlled current amplifier



$$Z_{ms} \cdot V = P_t (S_d - Bl\Theta)$$



The effective specific acoustic impedance at the diaphragm then reads :

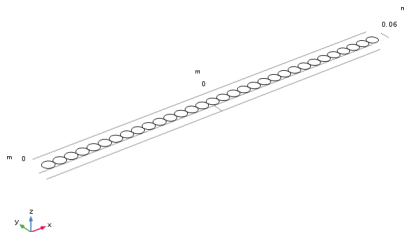
$$Z_a(\omega) = \frac{P_t(\omega)}{V(\omega)} = \frac{Z_{ms}(\omega)}{S_d - Bl\Theta(\omega)}$$

**Strategy :** to achieve a target specific acoustic impedance  $Z_{at}^{mn}(\omega)$  :

$$\Theta^{mn}(\omega) = \frac{S_d Z_{at}^{mn}(\omega) - Z_{ms}(\omega)}{Bl Z_{at}^{mn}(\omega)}$$

## Studied Active Metasurface geometry

To simplify the design, we will only consider a row of the 2D metasurface, made of  $M=32$  unit-cells ( $\phi_i = \phi_r = 0\text{rad}$ )



The  $m^{\text{th}}$  unit-cell reflection phase, at  $f_0$  shall be :

$$\Psi^m(f_0) = \Psi_0 - m \frac{2\pi f_0}{c_0} (2r_d)(\sin \theta_r + \sin \theta_i)$$

$$\text{with } \Psi_0 = \Psi^{M/2+1}(f_0) = -\pi$$

## Active metasurface design

- 1 choice of the target reflected angle  $\theta_r$  for a given incident angle  $\theta_i$  (at central frequency  $f_0$ ) ;
- 2 definition of the reflection phase grating  $\psi^m(f_0)$  over the metasurface of lattice constant  $2r_d$  according to  $\Psi^m(f_0) = \Psi_0 - m \frac{2\pi f_0}{c_0} (2r_d)(\sin \theta_r + \sin \theta_i)$  ;
- 3 definition of the reflection phase reference at the  $(M/2+1)^{\text{th}}$  cell such as  $\arg(\Gamma^{M/2+1}(f_0)) = -\pi$  ;
- 4 identification of the resonance shift  $\Delta f^m = f_0 - f^m$  for each cell over the metasurface, so that  $\arg(\Gamma^{M/2+1}(f^m)) = \psi^m$  ;
- 5 identification of the control parameters  $\mu_M^m, \mu_C^m$  achieving such resonance shift (with constant  $\mu_R = 1/3$ ) :

$$\bullet \mu_C^m = \frac{1}{2\pi Z_c S_d C_{ms} Q_s} \frac{1}{(f_0 + \Delta f^m) \mu_R}$$

$$\bullet \mu_M^m = \frac{Z_c Q_s S_d}{2\pi C_{ms}} \frac{\mu_R}{(f_0 + \Delta f^m)}$$

- 6 modification of the acoustic impedance of the  $m^{\text{th}}$  cell with the controller

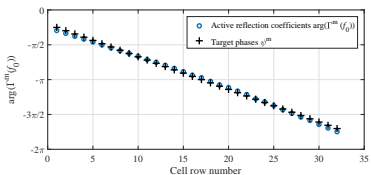
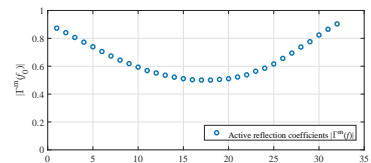
$$\Theta_t^m(\omega) = \frac{S_d}{Bl} \frac{(j\omega)^2 M_{ms} (\mu_M^m - 1) + j\omega (\mu_R S_d Z_c - R_{ms}) + \left( \frac{1 - \mu_C^m}{\mu_C^m C_{ms}} \right)}{(j\omega)^2 \mu_M^m M_{ms} + j\omega \mu_R S_d Z_c + \frac{1}{\mu_C^m C_{ms}}}$$



## Assessment of the achieved reflection coefficients

Let's consider the case where  $\theta_i = -\frac{\pi}{4}$  and  $\theta_r = \frac{\pi}{3}$ , and a metasurface composed of 32 unit-cells.

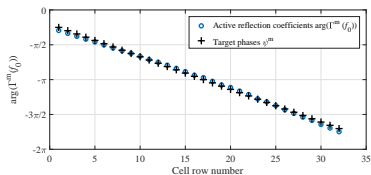
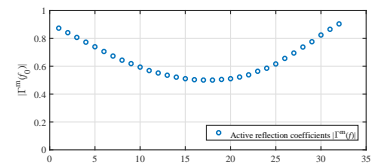
The target reflection phase, at  $f_0 = 343\text{Hz}$  ( $> f_s$ ), over the 32 unit-cells are :



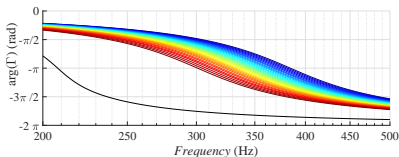
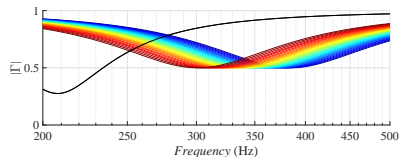
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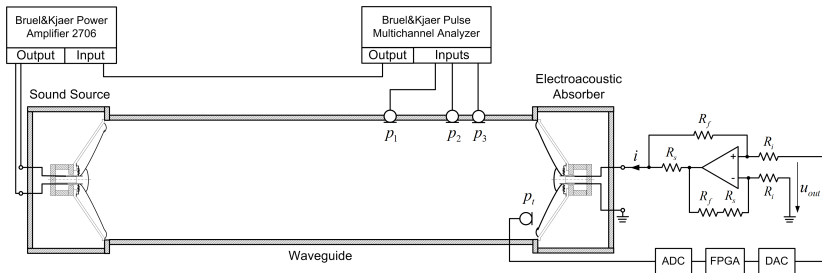


Corresponding reflection coefficient  $\Gamma^m(f)$  with corresponding values of  $\mu_M^m$  and  $\mu_C^m$  :



## Assessment of the achieved reflection coefficients

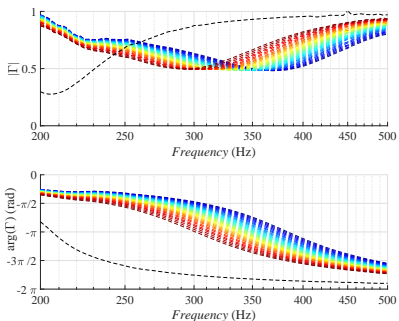
These settings are achieved experimentally on a MONACOR SPX-30M loudspeaker, and the reflection coefficient for each control case is assessed in an impedance tube :



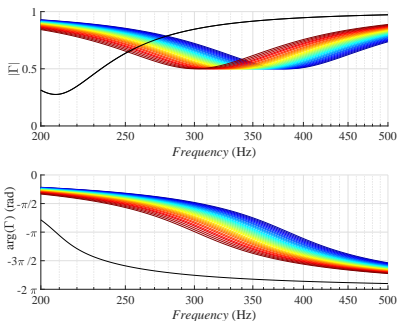
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Measurements



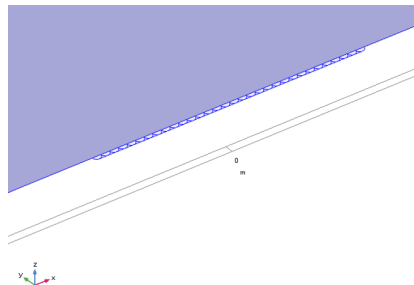
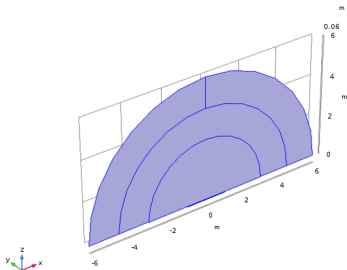
Simulations



## Simulation setting

The active metasurface simulation is performed with COMSOL Multiphysics with the Acoustics Module :

- 3D space dimension
- Background plane wave condition, with  $\theta_i = -\frac{\pi}{4}$
- cylindrical acoustic domain of radius 6 m, including 1.2m of PML, and a height of  $2r_d=6.4$  cm
- 32 unit-cells modelled as Acoustic Impedances with  $\mu_M^m, \mu_R$  and  $\mu_C^m$ .
- all other surfaces are modelled as "Sound Hard Boundary" (including the two delimiting  $xoz$  planes)
- Meshing : maximum element size =  $\lambda/6$  at 500 Hz.

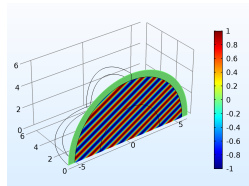


## Sound pressure fields

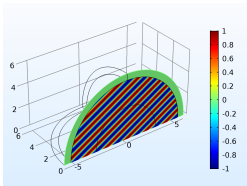
Two simulations cases :

$$\theta_i = -\frac{\pi}{4} \text{ and } \theta_r = \frac{\pi}{3}$$

$$\theta_i = -\frac{\pi}{4} \text{ and } \theta_r = 0 \text{ rad.}$$



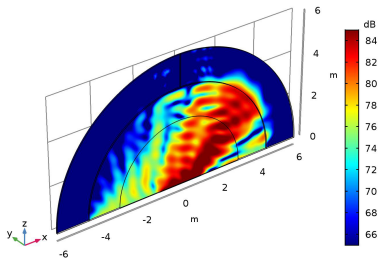
# Sound pressure fields



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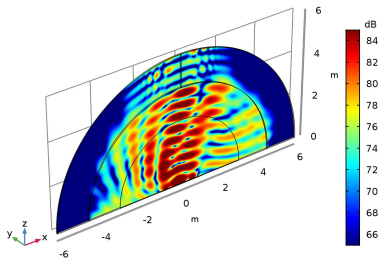
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f=350 Hz - Reflected Sound Pressure Levels Map (dB re. 20 μPa)



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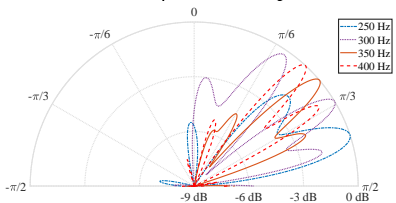
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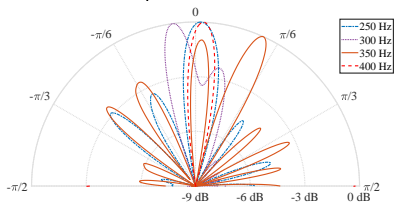
# Directivities

Two simulations cases :

$$\theta_i = -\frac{\pi}{4} \text{ and } \theta_r = \frac{\pi}{3}$$



$$\theta_i = -\frac{\pi}{4} \text{ and } \theta_r = 0 \text{ rad.}$$





## Conclusions

- Active Electroacoustic Resonators allow steering reflected wavefronts in a prescribed manner
- Effective at a central frequency, with relative bandwidth extension (up to one octave)
- Reflected coefficient higher than 0.5
- However, it is not yet possible to scan the full range of reflection phases ( $[-2\pi - 0]$ )

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### Toward wideband steerable acoustic metasurfaces with arrays of active electroacoustic resonators

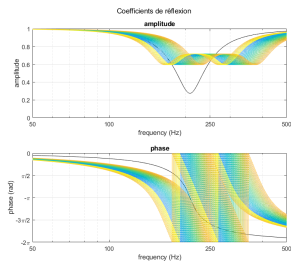
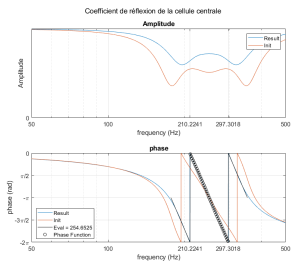
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# Perspectives

- Toward broadband reflection properties :  
Design Multiple Degrees of Freedom Electroacoustic Resonators (MDOF, instead of SDOF)



- Realization of an experimental (1D or 2D) prototype for validation of the reflection properties.
- Lowering the losses should allow practical application of the concept.